

Internal Ball Return UltraSpeed Ball Return Nominal Diameter 16-100 mm



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The production plant in Albstadt / Germany. Together with sister companies in Suhl and Dresden, the Steinmeyer group represents one of the leading makers of ballscrews and precision gauging instruments.



The company *Steinmeyer* was established in 1920, and was originally concerned exclusively with the production of precision measuring instruments.

Almost 40 years ago, a second line was launched with the development of what was then an almost unknown technology: the ball screw and nut assembly, now the company's most important product line.

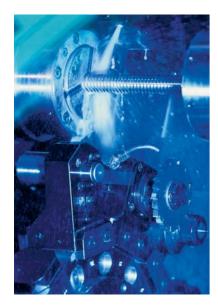
Steinmeyer is seen as one of the pioneers in this sector. Ballscrews were produced largely in those early days with nominal diameters between 5 and 16 mm for instrument engineering and for research equipment applications. With the advent of numerical controls in the machine tools sector, the production range was successively extended to take in larger sizes.

Today, *Steinmeyer* offers a wide product program ranging from 3 to 100 mm in diameter which sets whole new standards of quality. Experience and flexibility, as well as continually updated production facilities have made the name *Steinmeyer* synonymous with quality and reliability the world over.

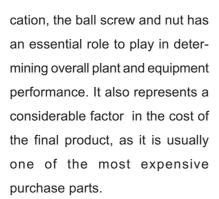
The company's continued expansion reflects the success of our close cooperation with our customers.



Up to Date with Steinmeyer



Ball screw and nut technology is not a new field in the world of industry. However, the systemrelated properties of the ball screw and nut such as minimal friction, preloading capability and nonstickslip effects, to name but a few, are nowadays no longer sufficient as such to equip today's high-tech machines and plants with adequate transmission systems. In practically every case of appli-



Steinmeyer has gained an outstanding reputation in this field. We supply top quality components





at internationally competitive prices to the most renowned names in the mechanical engineering industry. In cooperation with our customers, we provide individual solutions for a widely varying range of applications. Highly developed production engineering and our many decades of experience in this field ensure that we stay technically and economically up to date.

Steinmeyer flexibility and the scope of our production range are unique in the field. Our production program includes a standard program of miniature ballscrews in the range of 3 mm to 16 mm nominal diameter, and different shapes of standardized nuts for the range of 16 mm to 100 mm. Screw journals for this part of the program are always machined in accordance to our customers' drawings. Most commonly used flange nuts according to the ISO standard are part of our program as well as cylindrical nuts with fitting keys. Also, we feature a range of flanged nuts in combination with inch leads.



too.



As our nut design allows us to meet almost any possible requirement regarding nut outside dimensions, we can provide our customers with customized nuts,

Nonstandard nuts still use our unique design features, such as our sophisticated ball return and preloading systems. Thus our customers will benefit from the outstanding performance of our ballscrews without changing the design of their machines.

Our understanding is, that quality is a never ending process. This is why we seek proximity to our customers.

Quality means to us, that we use our knowledge to transform the needs of our customers into an individual solution to meet even the most demanding specifications - not less, but sometimes even a bit more.



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Symbols

- A: Screw cross section [mm²]
- α : Coefficient of thermal expansion [1/ °C]
- B₁₀: Nominal service life [10⁶ inches of travel]
- c: Specified lead deviation
- C_{0a}: Static axial load capacity [N]
- C_a: Dynamic axial load capacity [N]
- ΔI_p : Elongation [mm]
- ΔI_T : Thermal expansion [mm]
- d_{N} : Nominal diameter of screw [mm]
- D_N : D_N -value
- Δt : Temperature gradient [°C]
- "E": Perm. deviation for travel compensation (JIS)
- E: Young's modulus [N/mm²]
- ε: Extension
- e_{0a} : Mean lead deviation at the length I_u
- $e_{2\pi}$: Variation of lead within one revolution (JIS)
- e_{300} : Variation of lead within 300 mm of travel (JIS)
- e_p: Perm. deviation for travel compensation
- $e_{\mbox{\tiny sa}}$: Mean lead deviation at the length $\mbox{I}_{\mbox{\tiny s}}$
- k: Coefficient of bearing configuration
- F: Actual load [N]
- F_i*: Modified actual load [N]
- F_m: Dynamic equivalent axial load [N]
- F_m*: Modified dynamic equivalent axial load [N]
- F_{m1}: Dynamic equivalent axial load [LBS]
- F_{pr} : Preload [N]

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Numbering System	31
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Please see also our web page at	

http://www.steinmeyer.com for latest changes and news.

- F_{T} : Screw pretension force [N]
- i: Number of circuits
- JIS: Japan Industry Standard
- L₁₀: Nominal service life [10⁶ rev.]
- I_s : Length of unsupported shaft [mm]
- I_u: Full travel
- m: Coefficient of bearing configuration
- ni: Actual speed [rpm]
- nk: Critical speed [rpm]
- n_m: Average speed [rpm]
- n_{max}: Maximum speed [rpm]
- P: Lead [mm]
- P_{B} : Buckling load [N]
- P_i: Dynamic axial load capacity [LBS]
- q_i: Time of each duty cyclus [%]
- $R_{\text{b}} : \qquad \text{Axial stiffness of thrust bearing } [\text{N} / \mu \text{m}]$
- $R_{nu,ar}\!\!: \quad \text{Reduced stiffness of nut } [N/\mu m]$
- R_s : Stiffness of screw [N/µm]
- R_t : Total stiffness of ballscrew/bearing unit [N/µm]
- T: Specified lead deviation (JIS)
- T_{pr}: Preloaded idling torque [Ncm]
- $v_{2\pi a}$: Variation of lead within one revolution
- $v_{2\pi p}$: Perm. variation of lead within one revolution
- v_{300a}: Variation of lead within 300 mm of travel
- $v_{\scriptscriptstyle 300p}\!\!:$ $\,$ Perm. variation of lead within 300 mm of travel
- v_{ua}: Variation of lead over full travel
- v_{up} : Perm. variation of lead over full travel

As ballscrews are an indispensable element in today's machinery and apparatus construction, *Steinmeyer* has developed a wide program of standardized ball nuts for various purposes. This catalog shows standardized sizes from a nominal diameter of 16 mm through 100 mm. For miniature ballscrews with nominal diameters ranging from 3 to 16 mm, a separate catalog is available.

We edited this catalog to best suit the demands of the users, hence it is divided into two sections, providing you with a broad selection of technical background and design hints, and a listing of standard ball nut designs, respectively. This catalog is not intended to include all possible designs, and *Steinmeyer* will design any reasonable ballscrew according to your needs. Our engineering service is available to users worldwide through our distribution partners, and we shall be grateful to discuss your needs to provide you with the best ballscrew solution available.

Ballscrews provide power transmission from rotational to linear motion, or vice versa. Their main properties are high efficiency, durability and high precision. *Steinmeyer* has improved the basic design by introducing many detail solutions to improve performance of this product. As the needs in modern industries are increased speeds, reliability and precision along with better economics, the design in detail has changed reasonably. In general, every ballscrew shows an efficiency of approx. 90%, due to rolling ball contact between screw and nut. However, the demands today are more complex than simply transforming a rotational movement into a linear movement.

Steinmeyer ballscrews use a very complex profile of thread grooves of the screw and inside the nut. It is designed to meet requirements of increased stiffness, smooth motion and low friction.

These profiles, developed from the so-called **"gothic arch" profile**, allow for optimum control of contact angle, track conformity and backlash of single nuts.

Steinmeyer uses different ball return systems, based on which system provides an optimum solution for the ball screw size and application. They are all flush mounted with no external parts protruding from the outer diameter of the nut. Ball deflectors, manufactured in a special, computer controlled milling process, provide smooth motion and less wear on balls than external systems do.

The extreme low profile of our ball return allows for nuts to be designed in very small envelopes. This enables the use of larger screw nominal diameters, maintaining the same nut outside diameter, compared with conventional tube return nuts.



Standard Dimensions

Preloading Systems

Standard Sizes

Table 1 Metric Sizes

Inch Sizes

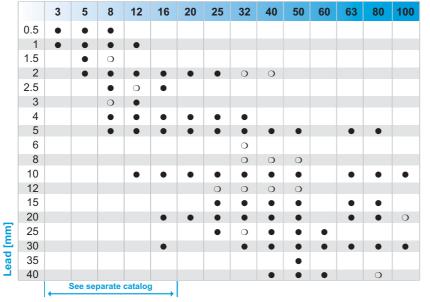
A large selection of inch sizes is available, too. However, they are not included in this catalog. Please inquire.

Maximum threaded length

Table 2

If your desired combination of diameter and lead is not listed here, please inquire. This shows only part of the program available. However, please select from this list, as these designs are often less expensive, and available with shorter lead time.

Nominal Diameter [mm]





O Please inquire

Nominal diameter	Recommended max. three	eaded length[mm]
[mm]	Accuracy grade 0 - 5	Accuracy grade 7 and 10
3	100 (150)	-
5	150 (250)	500
8	300 (450)	800
12	450 (700)	1200
16	650 (1000)	1600
20	800 (1200)	2000
25	1000 (1500)	2500
32	1300 (1900)	3000
40	1600 (2400)	_
50	2000 (3000)	-
60/63	2500 (3600)	_
80	3200	-
100	3500	_

Preloading Systems

Fig. 1

Fig. 2

Fig. 3.1

Fig. 3.2

Fia. 4

Steinmeyer uses four different systems for preloading to meet all requirements. Basically, preload serves to eliminate backlash and to increase stiffness. Since the preload also affects the friction torque and represents at the same time a load that has to be accounted for service life aspects, it needs to be controlled accurately.

Especially for miniature ballscrews Steinmeyer has developed a spring loaded double nut to meet most demanding requirements of low torque along with high accuracy in position. Please refer to our Miniature Ballscrew Catalog for details on our 1510 / 1530 series double nuts.

Single nuts are available either with backlash or preloaded by ball oversize. As preloaded single nuts always use a fourpoint contact between balls and ball race, the efficiency is reduced slightly compared to nuts with two-point contact. However, this design offers a good cost/performance ratio and allows economic solutions. The Steinmeyer ball return system, optimized for this purpose, does not need any spacer balls to improve smoothness. Hence, our preloaded single nut shows similar load capacities to double nut configurations.

For improved efficiency and stiffness the two-point contact offers optimum solutions. The lead offset needed to enable two-point contact can be achieved by shifting the grinding wheel a certain amount in axial direction in the middle of the nut's internal thread, dividing the nut into two parts where the balls are contacting on different sides of the balltrack. Both parts of the nut can be equipped with three or more full ballcircles according to the desired load capacity.

Steinmeyer UltraSpeed nuts are available with ball oversize preload (see Fig. 2). As an additional option for all such nuts with dual start threads, they can be supplied with a pitch offset between the two threads, enabling 2-point contact just like in a conventional double nut. However, these nuts are significantly more compact than 2-piece double nuts or lead offset nuts and provide increased stiffness.

The patented Steinmeyer double nut configuration with UNILOCK preloading system uses two nuts directly preloaded against each other and locked by means of an integral bonding system, avoiding the need of adjusting the distance between both nuts using shims. UNILOCK allows for reduced envelope double nuts with increased stiffness of the nut body and improved precision due to absence of extra parts. The factory set preload is maintained securely, along with a perfect alignment of the nuts. The UNILOCK system has proven its reliability in thousands of applications.

For accuracy grades 0 to 5, the values shown represent the maximum lengths for screws with preloaded single nuts. The values in parenthesis can be used, if larger-than-standard preload torque tolerances (see page 10) are acceptable, or if the ballscrew will be equipped with a double nut.

If your required length exceeds these values, please contact our applications engineering department for recommendations or for a detailed specification in this individual case.





Accuracy Grades

Accuracy Grades

DIN / ISO / JIS or ANSI Standard

Standard Lead Accurancy Grades

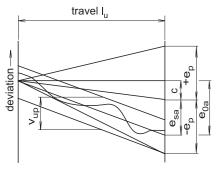
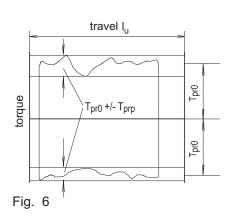


Fig. 5

Tolerance of Torque

Table 3



All designations used in this catalog fully comply to the German DIN 69 051/ part I-VI, and the European ISO/DIS 3408 standard. This applies especially to the calculation of load capacity and stiffness. When comparing, be sure to interpret values stated according to ANSI B5.48 standard correctly. If uncertain, please contact our engineering service for assistance.

Steinmeyer ballscrews are available as a standard in five different accuracy grades, according to the ISO standard (grades 1, 3, 5 for high precision ballscrews and grades 7 and 10 for normal precision ballscrews). Two accuracy grades have been added to meet requirements as per JIS standard grades 0 and 2. Regarding lead accuracy, both standards use five different values to define limits for mean travel deviation, travel variations and specified travel deviation. The respective designations that refer to the JIS standard are given in parenthesis.

- c: Specified lead deviation, is used to compensate heat effects or if the screw shall be held under tension (T).
- e_p: Represents the deviation of a straight line, which is to be drawn as an average of the actual lead deviation over the full travel (E).
- v_{up}: Permissible variation of lead over full travel, which is defined as vertical distance of two straight lines parallel to the average (e_p), which include maximum and minimum of the lead graph (e).
- v_{300p} : Same as v_{up} , but refers to the maximum within any interval of 300 mm (e_{300}).
- $v_{2\pi a}$: Variation of lead within one revolution $(e_{2\pi})$.

Tolerances ΔT_{prp} for torque variations T_{pr0} for double nuts or pitch-shift nuts, testing speed 100 min ⁻¹ ΔT_{prp} in % of T_{pr0} for $I_u \le 40 \cdot d_0$								
T _{pr0} [Ncn	n]	Grad	e					
from	to	0	1	2	3	5	7	10
5	10	40	45	45	50	60	-	-
10	20	35	40	40	45	50	_	-
20	40	30	35	35	40	50	70	-
40	60	20	25	35	40	40	60	-
60	100	20	25	25	30	35	40	-
100	250	15	20	25	25	30	35	-
		ΔT_{prp}	, in % of	T _{pr0} for	l _u > 40 ·	d ₀		
T _{pr0} [Ncr	n]	Grad	e					
from	to	0	1	2	3	5	7	10
5	10	-	-	-	-	-	-	-
10	20	50	50	60	60	60	_	-
20	40	40	50	50	50	60	-	-
40	60	35	40	50	50	50	70	-
60	100	35	40	40	40	45	50	-
100	250	30	35	35	35	40	45	_

Reduced tolerances and other testing specifications, as well as tolerances for single nuts upon request

Fig. 6: The mean lead deviation e_{0a} with tolerances $\pm e_p$ at the length I_u .

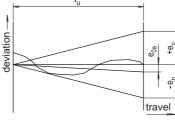


Table 3

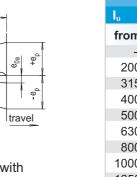


Fig. 7: The varation v_{ua} with tolerance v_{up} at the length I_{u} .

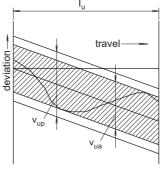


Fig. 8: The varation v_{300a} with tolerance v_{300p} at the length of 300 mm.

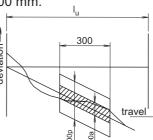
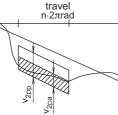


Fig. 9: The variation $v_{2\pi a}$ within 2π rad.(=1 rev.)



to
200
315
400
500
630
800
1000
1250
1600
2000
2500
3150

4000

3150

l _u	
from	to
_	200
200	315
315	400
400	500
500	630
630	800
800	1000
1000	1250
1250	1600
1600	2000
2000	2500
2500	3150

3150

4000

Tolerances v_{300p} for variations v_{300a} [µm]						
Grad	е					
0	1	2	3	5	7	10
4	6	8	12	23	52	210

Tole	rances	ν _{2πp} for v	variatio	ns v _{2πa} [um]	
Grad	le					
0	1	2	3	5	7	10
3	4	5	6	8	_	_



Tolerances e_p for mean lead deviation e_{0a} [µm]							
Gra	de						
0	1	2	3	5	7	10	
3	5	7	10	20	48	190	
4	6	8	12	23	52	210	
5	7	9	13	25	57	230	
6	8	10	15	27	63	250	
6	9	11	16	30	70	280	
7	10	13	18	35	80	320	
8	11	15	21	40	90	360	
9	13	18	24	46	105	420	
11	15	21	29	54	125	500	
-	18	25	35	65	150	600	
_	22	30	41	77	175	700	
_	26	36	50	93	210	860	
_	32	44	62	115	260	1050	

Tole	Tolerances v_{up} for variation v_{ua} [µm]					
Grad	de					
0	1	2	3	5	7	10
3	5	7	10	20	—	-
4	6	8	12	23	_	_
4	6	8	12	25	_	_
4	7	8	13	26	_	_
4	7	8	14	29	_	-
5	8	9	16	31	_	_
6	9	10	17	35	_	_
6	10	11	19	39	_	_
7	11	13	22	44	_	_
-	13	15	25	51	_	-
-	15	18	29	59	_	-
-	17	21	34	69	_	_
_	21	25	41	82	_	_

Accuracy Grades

Accuracy Grades

Quality inspection	ality inspection Steinmeyer uses an in-factory quality management system which uses the latest national and international standards available. For full documentation		Table 3.2				
	and traceability, every ballscrew can be supplied as an option with test records regarding:	d traceability, every ballscrew can be supplied as an option with test					
			d _N [mm]	Grade			
	1. Lead accuracy 2. Preload drag torque		over to	0 1 2 3 5 7 10			
	3. Stiffness		16 20	4 6 8 9 12 15 18			
	5. Sumess		20 32	5 7 9 10 13 16 19			
	Since Steinmeyer is a manufacturer of high precision gauging instruments		32 50 50 80	6 8 11 12 15 18 21 7 9 12 13 17 20 23			
	as well, our environmentally controlled gauging rooms are equipped with the		50 80	7 9 12 13 17 20 23			
	Every ballscrew with 20 mm nominal diameter and larger will be marked with a unique series number to ensure full traceability of the product.	Perpendicularity t ₄	Table 3.3 nominal dia.	perpendicularity tolerance t₄ [µm]			
Tolerances of Run-out			d _N [mm]	Grade			
and Perpendicularity			over to	0 1 2 3 5 7 10			
			16 20	2 3 3 4 5 7 9			
Fig. 10			20 32	2 3 4 4 5 7 9			
			32 50	2 3 4 4 5 7 9			
			50 80	3 4 5 5 7 10 13			
	$ \begin{array}{c c} \hline \\ \hline $						
		Design of the second seco					
		Perpendicularity t ₅	Table 3.4				

The method of supporting the ballscrew during measuring should ensure that the values taken are free of influences of a possible bending of the screw due to its weight. Generally supporting the screw between centres is appropriate. However, for very long screws it is recommended to take care for such influences and choose a different method.

The values listed below represent general rules. For distinct applications tolerances may vary.

Table 3.1								
nominal	dia.	run	out tole	erance t	2 [µm]			
d _N [mm]		Gra	de					
over	to	0	1	2	3	5	7	10
16	20	5	7	9	10	13	16	19
20	32	6	8	10	11	14	17	20
32	50	7	9	12	13	16	19	22
50	80	8	10	13	14	18	20	24

Run-out t₂

nomina	l dia.	run-	out tole	erance t	2 [µm]			
d _N [mm]		Gra	de					
over	to	0	1	2	3	5	7	10
16	20	5	7	9	10	13	16	19
20	32	6	8	10	11	14	17	20
32	50	7	9	12	13	16	19	22
50	80	8	10	13	14	18	20	24



ia.	perp	pendicu	larity to	lerance	t₅ [µm]							
	Grad	de										
to	0	1	2	3	5	7	10					
20	7	8	9	10	12	15	18					
32	7	8	9	10	12	15	18					
50	8	9	10	10	13	16	19					
80	9	10	11	12	15	18	21					

d_N [mm]

over

16

20

32

50

run-	out tole	rance t _e	, [μm]			
Grad	le					
0	1	2	3	5	7	10
5	6	7	8	10	13	16
5	6	7	8	10	13	16
6	7	8	8	11	14	17
7	8	9	10	13	16	19

Lifetime calculation

Lifetime calculation

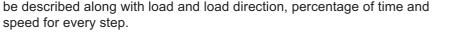
Load Capacity Selection

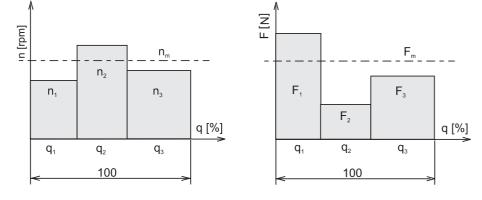
Fig. 11

Ballscrews usually will be used carrying axial loads under dynamic conditions. The selection therefore has to take into consideration the load and the travel - or number of revolutions - made under this load. The normal service life expectancy is based on the fatigue of the material of the balls.

Basically, travel made under higher load will determine the actual service life more than travel made under lower loads. As any application will give a constant load, a mean load must be calculated, which will result in the same service life. This so-called dynamic equivalent axial load F_m is then to be compared with the dynamic axial load capacity C_a.

For simplification, a typical work cycle of the machine under design should





In the simplest case-non-preloaded single nut - these values can be converted to the dynamic equivalent axial load F_m and the average speed n_m by means of the following Equations:

$$F_{m} = \sqrt[3]{\frac{q_{1} \cdot n_{1} \cdot F_{1}^{3} + q_{2} \cdot n_{2} \cdot F_{2}^{3} + \dots + q_{n} \cdot n_{n} \cdot F_{n}^{3}}{q_{1} \cdot n_{1} + q_{2} \cdot n_{2} + \dots + q_{n} \cdot n_{n}}} F_{m} = Dynamic equivalent axial load [N]}$$

$$F_{i} = Actual load [N]$$

$$F_{i} = Actual load [N]$$

$$n_{i} = Actual speed [rpm]$$

$$q_{i} = Time of each duty cycle [%]$$

$$n_{m} = Average speed [rpm]$$

In all other cases the influence of the preload must be accounted for by calculating the modified dynamic equivalent axial load F_m*. In case of single nuts preloaded by ball oversize approximate values can be calculated by means of following approximation:

Equation 2

Equation 1.1

Equation 1.2

Single nut with preload

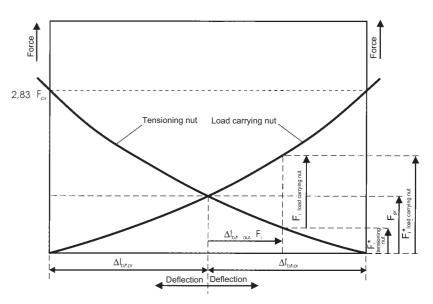
Double nut or pitch-shift nut Fig. 12: Preload Graph

Equation 3

Equation 4.1

Equation 4.2

Equation 4.3



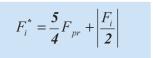
range.

To evaluate the resulting axial forces a computer program is needed. However, approximate values can be calculated by means of the following rule of thumb:

$$F_i^* = F_{pr} + \frac{F_{pr}}{2}$$



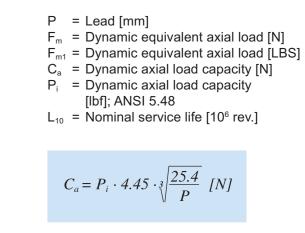






The preload graph demonstrates, how external loads cause a load increase in one nut, and a load decrease in the other one. At an external load of 2.83 times the preload, the tensioning nut is released completely. Even though this may happen without damaging the ballscrew, in most applications the preload should be set in order to keep all external loads within the preload

The resulting axial loads F_i* now can be used for calculating the dynamic equivalent axial load F_m^* by means of Equation 1.1. The load capacity for a ballscrew listed in the catalog is based on the ISO 3408 / DIN 69051 calculations. This dynamic load capacity is the axial load F_m*, under which the ballscrew will show a service life of 1 million revolutions (L₁₀ rating). To compare the ISO 3408/DIN 69051 with the ANSI 5.48 1977, load capacity C_a and service life must be converted in P_i and B₁₀ rating.



Lifetime calculation

Lifetime calculation Stiffness

Example of Lond	The following duty and is given for a marking.		The desired service
Example of Load Capacity Selection	The following duty cycle is given for a machine: q_i n_i F_i		The desired service percentage of 50 %. revolutions for the de
	q_i n_i F_i		
	1: 20% 250 rpm 2500 N >>>		$L_{10} = 20\ 000\ \mathrm{h} \cdot 0.5$
	2: 50% 1200 rpm 1600 N >>>		10
	3: 15% 2000 rpm 800 N >>>		By means of Equation
	4: 15% 2000 rpm 800 N <<<		
	For stiffness researce a bellearow with double put shall be calested		$C_{areq.} = F_{m}^{*} \cdot (L_{10} / 10)$
	For stiffness reasons a ballscrew with double nut shall be selected, and therefore the direction of the external load is to be considered.		A ballscrew with a d selected, resulting ir
	The preload is set to 1000 N to keep all axial loads within the preload		selected, resulting in
	range. As a rule of thumb, the following Equation can be used:		$L_{10actual} = (14600 / 15)$
Equation 5	$F_{pr} \geq 0.4 \cdot F_{imax}$		The resulting actual
	Both nuts must be calculated separately:		
	1: F_{A1}^{*} = 1000 N + 2500 N /2 = 2250 N 2: F_{A2}^{*} = 1000 N + 1600 N /2 = 1800 N 3: F_{A3}^{*} = 1000 N + 800 N /2 = 1400 N 4: F_{A3}^{*} = 1000 N = 200 N /2 = 600 N		It is not recommender range.
	4: F_{A4}^* = 1000 N - 800 N /2 = 600 N	Radial Loads	Ballscrews are desig
	The same applies to the second nut, but with inverted external loads:		catalog apply only to As there are always
	1: F _{B1} * = 1000 N - 2500 N /2 = (-250 N) ==> 0		guideways, there ma minimized. Under no
	Although Equation 3 leads to a negative result, this will simply lead to unloading of the second nut. Exact calculation or crosscheck with Equation 5 can ensure that this will not happen.		minimum axial load When considering a Steinmeyer enginee
	2. E * - 1000 N 1600 N /2 - 200 N		
	2: F_{B2}^{*} = 1000 N - 1600 N /2 = 200 N 3: F_{B3}^{*} = 1000 N - 800 N /2 = 600 N 4: F_{B4}^{*} = 1000 N + 800 N /2 = 1400 N , NUT B "	Stiffness	Besides the pure ge influenced by the sti <i>Steinmeyer</i> describ
	As the resulting loads in nut "A" are reasonably higher and the nut shall be symmetrical (both nuts same load capacity), the service life of the ballscrew will be determined by nut "A".		and uses the same designations:
	of the ballscrew will be determined by hut "A.		R _{b/t} Stiffness of b
	By means of Equation 1.1 the equivalent dynamic axial load in nut "A" is:		in balls and b
	F _m * = 1583 N		R _{nu} Stiffness of r deformation
	The average speed is:		
	n _m = 1250 rpm		R _{nu,ar} For single nu R _{nu,ar} than for ded nuts are



ce life shall be 20000 hours (L_h) with a working %. With the average speed, the number of e desired service life can be evaluated:

.5 • 60 min/h • 1 250 • rpm = 750 • 10⁶ rev.

ation 4.2 the required dynamic axial load capacity is:

10⁶)^{1/3} [N] = 14380 N

a dynamic axial load capacity of 14600 N (4/25 mm) is g in a service life of:

1583)³ • 10⁶ rev. = 784 • 10⁶ rev.

al service life expectancy should be in the range of:

 $10^6 \leq L_{10} \leq 10^9$

nded to rely on service life expectancies outside the above

signed to take axial loads. The load capacities given in this / to pure axial loading.

ys tolerances in the alignment of bearings and linear may be a small amount of radial force, which should be normal conditions, a radial load less than 5% of the ad will not cause any problems.

g a ballscrew for use under radial load, please consult neers.

geometric accuracy the precision in position is mainly stiffness (rigidity) of a ballscrew drive.

ribes the stiffness in the manner of the DIN / ISO standard the designations: This catalog uses therefore the following

of ball contact zone: Includes all deformations incorporated d ballrace

of nut: Deformations described under $R_{b/t'}$ plus the on of the nut

nuts without preload the catalog gives different values for for the preloaded version. Generally stiffnesses for preloaare higher than for non-preloaded ones.

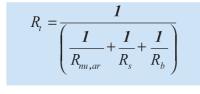
Stiffness

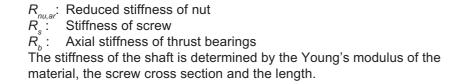
As an approximate estimation, twice the stiffness of a non-preloaded nut can be obtained by preloading the nut.

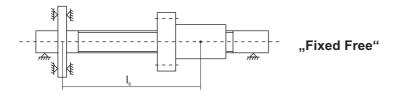
The values listed in the catalog apply only to deflections of the nut. For total stiffness of the drive the stiffnesses of the shaft under load and the stiffness of the thrust bearing must be taken into account. The total stiffness (R) is calculated as follows:

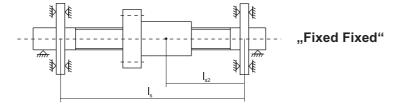
Equation 6

Fig. 13









When using the fixed-free mounting method, the stiffness of the shaft is:

$$R_{sI} = A \cdot \frac{E}{l_s} \cdot \mathbf{10}^{-3}$$

Equation 7

In case of **fixed-fixed** mounting method it is:

$$R_{s2} = \mathbf{2} \cdot A \cdot \frac{E}{l_{s2}} \cdot \mathbf{10^{-3}}$$

with $I_{2} = 0.5 \cdot I_{2}$ $E = 210000 \text{ N/mm}^2$ $R_{s2min} = 2 \cdot A \cdot E / I_{s2} \cdot 10^{-3}$ Table 5

A = Screw cross-section [mm²]

d _N P	16	20	25	32	40	50	60	63	80	100
2	175	282	450	_	_	-	_	_	-	_
4	151	251	411	701	_	_	_	_	_	_
5	144	241	398	685	1106	1774	_	2877	-	-
10	144	241	398	605	946	1569	_	2615	4382	7043
15	-	-	398	605	946	1569	-	2519	-	-
20	_	241	398	605	1004	1495	_	2395	4097	-
25	-	-	_	605	1008	1569	2260	_	_	-
30	_	_	_	605	_	1569	2260	2395	4097	6680
35	-	_	_	_	_	1569	_	—	-	-
40	_	-	-	_	1004	1569	2260	_	_	_

Example

ISO-standard grade 3, System: fixed-free $R_{nu, ar} = 490 \text{ N/}\mu\text{m},$ Bearing INA ZARN 2062 LTN,

$$A = 605 \text{ mm}^{2}$$

$$I_{s} = 1000 \text{ mm}$$

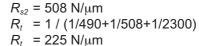
$$R_{s1} = (A \cdot E) / (I_{s} \cdot E)^{2}$$

$$R_{s1} = 127 \text{ N/}\mu\text{m}$$

$$R_{t} = 1 / (1/R_{nu,a} - R_{t} = 1 / (1/490 + 1)^{2}$$

$$R_{t} = 97 \text{ N/}\mu\text{m}$$

used:



Equation 8



Ballscrew 10 x 32 mm, double nut with 2 x 3 circuits Preloaded to 10% of the dynamic load capacity

Stiffness R_b = 2300 N/µm (see page 29)

10³)

 $+ 1/R_{s} + 1/R_{b}$ 1/127+1/2300)

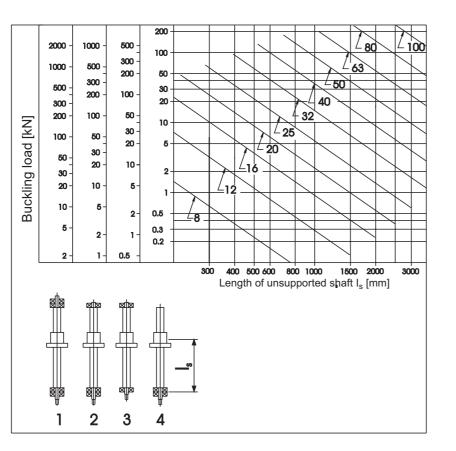
For improved stiffness of the axis, a fixed-fixed bearing configuration can be

Buckling load

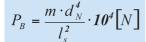
Critical Speed

Critical Column Load

Table 6



For quickly determining the buckling load, same result can be obtained by calculating it by means of the following Equation:



- P_{B} = Buckling load [N]
- d_N = Nominal diameter of ballscrew [mm]
- = Length of unsupported shaft [mm] I_s

т	= Coefficient of bearing cor	nfigura	tion (see table 6)
	fixed - fixed:	22.4	(1)
	fixed - supported:	11.2	(2)
	supported - supported:	5.6	(3)
	fixed - free:	1.4	(4)

For safety reasons, a factor of 0.5 must be applied:

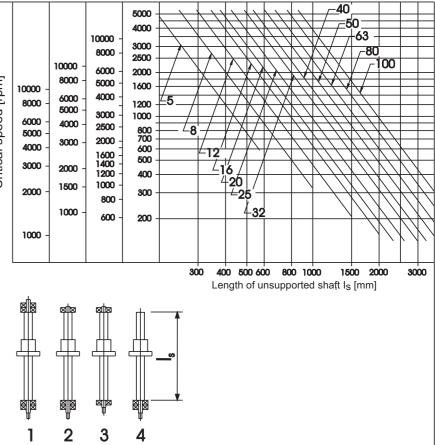
Equation	10
----------	----

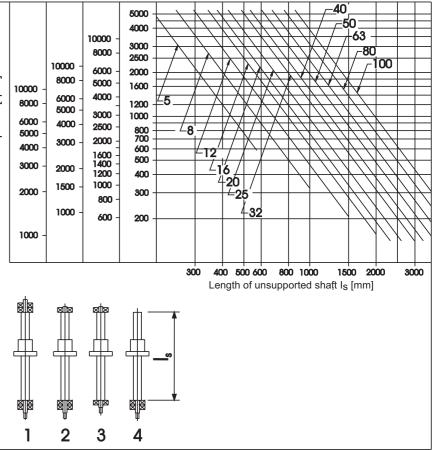
Equation 9

 $F_{max} = 0.5 \cdot P_B$

Critical Speeds	reson For ro speed	ritical s ance. otating d accor nal dia
Table 7	Critical speed [rpm]	10000 - 8000 - 6000 - 5000 - 4000 - 3000 - 2000 - 1000 -

Equation 11





For quickly determining the critical speed, the same result can be obtained by calculating it by means of the following equation:

 n_k =

- n_{ν} = Critical speed [rpm]

- - fixed fixed: fixed - supported:
 - fixed free:



tical speed is the speed, where the screw starts vibrating because of

ating screws the maximum speed (rpm) is determined by the critical according to the graph below. This speed depends on the screw's I diameter, length and bearing method.

$$= k \cdot d_N \cdot \frac{1}{l_s^2} \cdot 10^7 [1/\min]$$

 d_{M} = Nominal diameter of ballscrew [mm] = Length of unsupported shaft [mm] \vec{k} = Coefficient of bearing configuration (see table 7) 25.5 (1) 17.7 (2) supported - supported: 11.5 (3) 3.9 (4)

Critical speed

Lubrication

I limitation is alls are not ru but have to f ion, the max he lubricant. nass of the b to the speed mum speeds ximum speed	s given by unning with follow the ximum spectrum balls and the d. d_N s refer to for ed [rpm]	ith constant defined is not the result $n_{max} \cdot a$	kimum sp ant spee lector gro ot simply ting acco	ed throug ooves ar / determi	gh the nu nd thus r ined by f	eceive a the temp	n axial erature	Oil Lubrication	When lubricating when lubricating when lubricating when some of the oil during start-up and lubrication film, the during conditions of HLP, or high-preserved.
alls are not re but have to f ion, the max he lubricant. hass of the b to the speed	s refer to f ed [rpm]	ith constant defined is not the result $n_{max} \cdot a$	ant spee lector gro ot simply ting acco	ed throug ooves ar / determi	gh the nu nd thus r ined by f	eceive a the temp	n axial erature	Oil Lubrication	Eubrication is a co Failure to do so m application engine When lubricating v both ends. The oil during start-up and lubrication film, the during conditions of HLP, or high-press viscosity can be for
but have to f ion, the max he lubricant. nass of the b to the speed	follow the ximum spectrum spe	ball defl eed is not the result = $n_{max} \cdot a$	lector gro ot simply ting acco l_N	ooves ar / determi	nd thus r ined by f	eceive a the temp	n axial erature	Oil Lubrication	When lubricating weights both ends. The oil during start-up and lubrication film, the during conditions of HLP, or high-preserved.
nass of the b to the speed mum speeds	balls and t d. d_N s refer to f ed [rpm]	$= n_{max} \cdot a$	d_N	eleration	is and fo	orces put	а	Oil Lubrication	both ends. The oil during start-up and lubrication film, the during conditions of HLP, or high-press
	s refer to f ed [rpm]								lubrication film, the during conditions HLP, or high-pres
	ed [rpm]	following	g table:						viscosity can be fo
ximum spee									
									Ball screws to be u Isoflex [®] Dispersion
13 13 26	d _∾ -valu 120.00 120.00 160.00	00 00							lubrication and pro installation of the l installation. Once out.
20 25	32	40	50	60	63	80	100		
200 1800	_	_	_		_	-	-	Grease Lubrication	As an all-purpose
100 3600	2800	-	_		_	-	-		Normally, the ball a grease gun, or w
100 3800	3300	2600	2100		1700	1500	-		factory grease car
100 3800		3000	2500 2500		2000	1500	1200		through piping mo
							_		be used. Nuts sho
				2500	2000	-	_		www.steinmeyer.
	1000	3500	3000	2500	2000	2000	1600		and a list of appro
	-	-	3000	-	-	-	-		
		3500	3000	2500	-	-	-		
10	- 3800 00 4000 - 4000 	- 3800 3400 00 4000 4000 - 4000 - 4000	- 3800 3400 3000 0 4000 4000 3000 - 4000 - 4000 4000 3500 	- 3800 3400 3000 2500 00 4000 4000 3000 2500 - 4000 - 4000 3000 - - 4000 3500 3000 - - - 3000 3000	- 3800 3400 3000 2500 00 4000 4000 3000 2500 - 4000 - 4000 3000 2500 - - 4000 3500 3000 2500 - - 4000 3500 3000 2500 - - 4000 3500 3000 2500	- 3800 3400 3000 2500 2000 00 4000 4000 3000 2500 2000 - 4000 - 4000 3000 2500 - - - 4000 3500 3000 2500 2000 - - 4000 3500 3000 2500 2000 - - - 3000 - -	- 3800 3400 3000 2500 2000 - 00 4000 4000 3000 2500 2000 1500 - 4000 - 4000 3000 2500 - - - - 4000 3500 3000 2500 2000 2000 - - - 3000 2500 - - -	- 3800 3400 3000 2500 2000 - - 00 4000 4000 3000 2500 2000 1500 - - 4000 - 4000 3000 2500 - - - - - 4000 3500 3000 2500 0 - - - - 4000 3500 3000 2500 2000 1600 - - - 3000 - - - -	- 3800 3400 3000 2500 - - 00 4000 4000 3000 2500 1500 - - 4000 - 4000 3000 2500 - - - 4000 - 4000 3000 2500 - - - - - 4000 3500 2500 2000 1600 - - - - 3000 - - -

In order to avoid vibration problems due to exceeding the critical speed, the screw can be used stationary, while the nut is rotating.

Since *Steinmeyer's* ball return system is suitable for rotating nuts as well, same speeds can be obtained for rotating nuts. The maximum speeds listed in this catalog apply only to use of the ballscrew under normal conditions. Extreme accelerations, vibrations or permanent use at high speeds, which may lead to extensive heating and possible failure of the lubricant, may reduce service life. Please make use of the experience of our application engineers in such cases.

Lubrication Under Extreme Conditions

Extreme operating conditions, such as high accelerations, vibration, or prolonged operation at or near the maximum recommended speed may cause problems due to excessive heat generation. This in turn may have an adverse impact on the performance of the lubricant and possibly cause premature failure. In such cases, or when planning to use ball screws under unusual environmental conditions (below -20° C or above +100° C, under vacuum, radiation, clean room conditions, in aerospace applications, etc.), or with short, repetitive strokes, please contact our engineers.



an be lubricated with grease or oil. The selection of lubricants tion intervals and lubricant quantities depends on the ditions. As a general rule, ball screws are best lubricated with oil supply, except when speeds are very low and surface or when minimal lubricant loss or even for-life lubrication is

a complex subject. Always follow our lubrication guidelines. so may void the warranty. If uncertain, please contact our igineering service.

ing with oil, the ball nut should have an oil port and wipers at the oil used must withstand very high pressure. As speeds of and reversing are insufficient to build a hydrodynamic in, the oil should have additives that prevent excessive wear ons of boundary friction. Such oil is generally rated as CLP or pressure gear oil. Guidelines for proper quantities and oil be found at our website *www.steinmeyer.com*.

be used with oil lubrication will be treated with Kluber rsion at the factory. This special grease guarantees proper d protection from corrosion during storage, inspection and the ball screw. It is not necessary to remove it before nce the oil supply is operational, the grease will be washed

bose grease we recommend Kluber Staburags[®] NBU 8 EP. ball screw will have to be re-lubricated, either manually using or with an automated grease supply. If necessary, the e can be substituted with a lighter grease that can be fed g more easily. Only grease with high-pressure rating should is should be fitted with wipers. Consult our website at eyer.com for recommended lubrication intervals, quantities pproved greases.

Lubrication

Lubrication

Re-Lubrication	Lubrication intervals depend mainly on the operating conditions. Generally, lubricant loss in ball screws is higher than in ball bearings. Covers or bellows that protect the ball screw from chips, dirt, coolant or aggressive vapors, as well as the proper selection of wiper seals (see page 25) play an important role in reliability and life. Accelerations and speeds may also have an impact. The following rules therefore can only provide general guidelines.	Automatic Lubrication	For au liquid g inform The re approy better
	For applications requiring for-life or long-term lubrication, Steinmeyer supplies ball screws fitted with grease packaged nuts and felt wipers. These felt wipers act as a lubricant reservoir and minimize lubricant loss.	Long-Term or For-Life Lubrication	It is po to the combin wipers grease
Manual Re-Lubrication	Lubrication intervals depend on the type of wipers used. The following recommendations assume that the ball screw is sufficiently protected from contamination with telescopic way covers, bellows or similar. Any direct exposure to chips, dirt and coolant should be avoided. Excessive contact with contaminants, solid or liquid, may cause reduced wiper life or make it necessary to clean and re-lubricate ball screws more frequently.		This sy plastic will ab from ir signific

Table 10

Wipers	Lubrication Interval
None	2 months or 300 hours
Labyrinth seals	3 months or 500 hours
Contact wiper	4 months or 650 hours
Felt or combination wipers	6 months or 1000 hours

Apply grease through the lube port into the nut. When a grease gun is used be careful not to overfill the nut. This may result in excessive heat generation. Combination or felt wipers may become damaged or dislocated if the nut is pressurized.

If no lube port is available, apply grease as an even layer along the screw thread.

If possible, any visible used grease or dirt should be wiped off prior to relubrication. Please refer to our website at *www.steinmeyer.com* for recommended grease quantities.

Wipers and Seals

Steinmeyer supplies four basic wiper designs:

For automatic grease lubrication, use grease with lower viscosity or even liquid grease. Refer to your grease pump manufacturer for viscosity information. Only use grease with EP-additives to reduce wear.

The recommended lubrication interval for automatic grease lubrication is approx. 8 hours. Generally, shorter intervals and smaller quantities are better than long intervals with high quantity.

It is possible to extend re-lubrication intervals to over 1000 hours, or even to the rated life of the ball screw. This is accomplished with our combination wipers, or, in a perfectly clean environment, with our felt wipers only. The nut will be pre-greased at the factory with a long-life grease, and the felt reservoir is saturated with lubricant.

This system offers significant advantages over other, oil-impregnated plastic wipers: The felt holds lubricant up to 70% of its own volume, and it will absorb any contaminant from outside the nut as well as wear particles from inside. Thus, the lubricant is kept clean, extending the lubricant life significantly. And, because in heavy-duty applications such as high speed machine tools it is still advisable to re-lubricate whenever possible, the felt reservoir offers the advantage of being able to re-absorb lubricant. There is no need to replace wipers, since they can be re-filled by pumping some lubricant into the nut. However, should they become saturated with debris, it is advisable to replace them. This can be done without dismantling the ball screw – another advantage of our combination wipers.

Wipers protect the ball nut from contamination. They also distribute lubricant and prevent lubricant loss in some cases. Manufactured from durable plastic material, they are compatible with all commonly used lubricants and resistant to most coolants. For certain applications in extreme environments, such as very high or very low temperatures or radiation, custom wipers made from PTFE or PEEK are available upon request.

/pe

Labyrinth se

Contact wip

Felt wiper

Combinatior wiper



	Application	Lubrication	Friction
eal	Standard	Oil or grease	Low
ber	Dirty environment	Oil or grease	Medium
	Long-term lubrication and clean environment	Oil or grease	High
n	Maximum protection against fine or abrasive dirt, longterm lubrication	Oil or grease	High

Lubrication

Lubrication

Labyrinth seals

These are the most frequently used wipers. Since they leave a gap about 0.2 mm above the screw thread, they should not be used in dirty environments. When used with automatic oil lubrication of the ball nut, they allow flushing of the nut with fresh lubricant. Oil consumption is relatively high, but friction is very low. Operating temperature range is from -20° C to +100° C.

Felt wipers

These are made of durable, dense felt and efficiently seal the nut. Lubricant loss is very low, and since felt absorbs and stores lubricant, felt wipers are suitable for long-term lubrication. They are able to protect the ball nut from very fine particles and even absorb wear particles generated inside the nut, thus keeping the lubricant clean. Felt wipers however have a limited life in dirty environments, and they should not be exposed to coolant or other water-based fluids. Maximum temperatures should not exceed +70° C. Friction is relatively high.

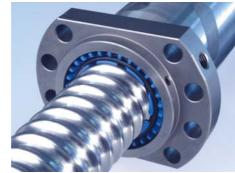


Segmented contact wipers

These wipers are made from durable plastic material, and several elastic segments maintain contact with and slide on the screw surface. A spring keeps this contact constant. The segments provide multiple wiper lips which makes these wipers very efficient in a dirty environment. An additional, uninterrupted wiper lip keeps the lubricant inside the ball nut. Segmented contact wipers have a somewhat elevated friction compared to labyrinth seals. Temperature range is from -20° C to +100° C.



Segmented wiper with protruding fingers



Segmented wiper, flush mounted

Combination wipers

Operation of

wipers

Materials

ball screws without

These combine advantages of the segmented contact wiper, plus those of the felt wiper. They provide optimum protection against large amounts of contaminants, including fine, abrasive dirt such as grinding sludge, paper or wood fibers, and dust. Since the felt wiper is inside the ball nut and behind the plastic wiper, it is well protected against fluids from outside. Because of the perfect sealing against contamination and the lubricant retention and storage capability of the felt, they are suitable for long-term lubrication, or even for-life lubrication. Steinmeyer combination wipers can be used with any liquid lubricant, grease or oil, and allow selection of the lubricant that best suits the application - an important advantage over oil-impregnated plastic wipers. They can be re-filled with lubricant in intervals of several hundred hours, if necessary. Used in conjunction with automatic oil or grease lubrication, they allow the lubricant flow to be significantly reduced.



carefully.

Steinmeyer ballscrews are made of high quality steel to ensure best performance and durability. Purity, hardness, absence of stresses and micro-cracks are most important. Steinmeyer uses steel suppliers, who have to conform to very restricted conditions.

to fatigue.

Hardnesses are:

nut: balls:



If the ball nut is not fitted with any wiper, it must be protected efficiently against any outside contamination. Frequent re-lubrication or automatic oil supply is necessary. Steinmeyer recommends reviewing such applications

Depending on the nominal diameter the screw is made of AISI 1050/1055 or AISI 52100. Nuts and balls are made of AISI 52100. Steel materials are vacuum-melted to reduce impurities, which inherently increases resistance

Ballraces are induction hardened, balls and nut are through hardened. Upon request, bearings journals will be hardened, too.

ballrace zone of screw: HRC 60 ± 2 (700 HV 10) HRC 62 ± 2 (740 HV 10) HRC 62 ± 2 (740 HV 10)

Lubrication

Bearings

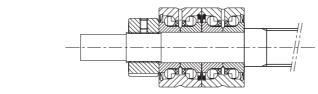
Special materials/ vacuum use	For special applications, <i>Steinmeyer</i> ballscrews are also available in corrosion resistant steel. For use in vacuum chambers, normal materials used may lead to outgassing. Special executions for such applications are available, please use our engineering service.	Bearings	As the axial stiffnest supporting the screw used. However, kee diameter must be sr Otherwise, it will be both ends will make cost upon repair.	w at both ends by p in mind, that at naller than the roo impossible to mo	thrust bearings, thi least on one end th ot diameter of the b unt the nut. Using s	is method is often ne maximum jou pallthread. shrunk bushings
Ballscrew Journal Design	Steinmeyer recommends using INA support bearings. Upon request, our engineers will make recommendations concerning which bearing to use and how to design the journals accordingly.		The following table However, it is not po refer to our enginee	ossible to cover al	l combinations in th	
	In machine tool applications, basically, only bearings providing a similar axial stiffness to the ballnut should be used. The size of the bearing is to be determined – for better cost effectiveness – according to the major (nominal) diameter of the screw, as this is generally used as a shoulder to preload the bearings pack (see fig. 14.1).		Ballscrew nominal	INA-bearing		
	bearings pack (see lig. 14.1).	Table 13	diameter [mm]	w/journal config	uration acc. to fig.	
	If the bearing's recommended minimum shoulder diameter comes close to			14.1	14.2	14.3
	the major diameter of the screw, the ballthread should not penetrate the		16	ZKLN1034	-	ZKLN1242
	shoulder in order to allow for a full 360° surface. This will prevent problems		20	ZKLN1242	-	ZKLN1545
	such as cocking or skewing (see fig. 14.2).		25	ZKLN1747	-	ZKLN2052
			32 (P ≤ 5)	ZKLN2557	-	_
	In cases of the bearing's recommended minimum support (shoulder)		32 (P ≤ 10)	ZKLN2052	ZKLN2557	-
	diameter being larger than the screw's nominal or outside diameter, we will		40 (P ≤ 5)	ZKLN3062	-	-
	provide a solid shoulder using a bushing shrunk on the journal, so that it can		40 (P ≤ 10)	-	ZARN3062LTN	-
	be ground to optimum perpendicularity. Please avoid – if possible – to		$50 (P \le 5)$	ZKLN4075	-	—
	design wrench flats on this diameter, as the ballscrew must be machined		50 (P ≤ 10)		ZARN4075LTN	-
	from larger diameter raw material, which may increase costs (see fig. 14.3).		$63 (P \le 5)$	ZKLN5090	_	-
			63 (P ≤ 10)	ZARN4090LTN		_
					ZARN45105LTN	
			80 100	-	ZARN43103LTN ZARN50110LTN ZARN60120LTN	_

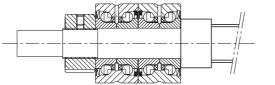
This brief overview cannot give a final selection aid to determine an optimum bearing solution. Radial loads due to drive belt tension or increased axial loads due to pre-tensioning a ballscrew need to be considered, too. Steinmeyer has CAD software available, which is used by our engineers to select the optimum solution for your individual application.



Fig. 14.2









v ften ournal igs at ktra

mblies. lease

Tensioned Ballscrews

Part II

Tensioning a Ballscrew	For eliminating loss of accuracy due to thermal expansion, the ballscrew can be held under tension. In this case the lead error can be specified to be negative ¹⁾ . Bearings must be selected accordingly. <i>Steinmeyer</i> will provide you with calculations regarding proper selection of bearings and resulting axial stiffness upon request. The elongation due to pretensioning must be greater than or equal to the thermal expansion that would occur under the maximum temperature increase expected.	Numbering systems	The <i>Steinmeyer</i> numbering system is divided in 1. The order number describing the entire 2. The item (or catalog) number describing For enquiries please use the type number listed make sure to provide us also with drawings of the with additional information regarding accuracy gr Once a customized design has been created, it w individual item number .
Equation 15	$\Delta l_p \geq \Delta l_T$ The pretensioning force must be considered carefully, as it will affect load capacity evaluations for the support bearings.	Item Number	1416.123/5.32.245.318 T3 P ISO-standard grade overall length of screw I ₀ [mm effective threaded length [mm]
Equation 16	$F_{T} = E \cdot A \cdot \varepsilon$ $\varepsilon = \frac{\Delta l_{p}}{l_{s}}$ F_{r} = Screw pretension force [N] E = Young's modulus [N/mm ²] $\varepsilon = \text{Extension}$ $\Delta l_{p} = \text{Elongation [mm]}$ $l_{s} = \text{Length of unsupported shaft [mm]}$ $A = \text{Screw cross section}^{2} \text{ [mm2]}$	Type Number	lead P [mm] Identification for custom versi type of nut 1416/4.20.3.3 Image: state st
Equation 17	The thermal expansion (ΔI_T) can be defined as follows: $\Delta I_T = I_s \cdot \Delta t \cdot \alpha$ $\alpha = \text{Coefficient of thermal expansion [11.5 \cdot 10^{-6} / ^{\circ}\text{C}]}$ $\Delta t = \text{Temperature gradient [}^{\circ}\text{C}\text{]}$	Extra ballcircles for increased load carrying	Pls note:Dimensions in item numbers, such as nominal di length, will be separated by periods.Decimals will be shown using a comma instead of point.Example: $1414/5,08.16.013$ Meaning: $d_N:16$ mm P: 5.08 mmIf you cannot find a nut to meet your load cap enquire also. Standard nuts as per this catalon
 Fig. 15 ¹⁾ see also specified lead deviation (page 10) ²⁾ see also cross section (table 5, page 19) 		capacity	nuts, are available with additional load carryin higher load carrying capabilities.

(table 5, page 19)



in two parts:

ire assembly bing the nut only

sted in part II. Please f the screws journals and y grade, preload etc. , it will be assigned an

8	ТЗ	Ρ	

appendix for backlash free execution
ISO-standard grade
 overall length of screw I ₀ [mm]
 effective threaded length [mm]
nominal diameter d _N [mm]
 lead P [mm]
 Identification for custom version
type of put

l dia., lead or ad of decimal

capacity requirement, please talog, as well as customized rrying ballcircles to provide

Nominal Diameter 16-20 mm

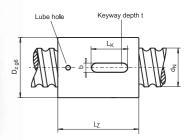
Single Nut with Ball Oversize Preload Precision Ground Execution Grade T0 - T5



	Tech	hnical Da	ta								Dime	ension	S												
	Nut	type	Lead	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating	stat. load rating	Stiffness •		Flan	ged nu	t with v	wipers	both e	nds ∎					Cylin with f	drical ı itting k	nut wit keyway	hout w	ipers
			Р	d _N	T	d _w	Ca	C _{0a}	R _{nu,ar}		LF		L ₁	D_4	D ₅	D_6	L ₇	L_3	н		Lz	Dz	b	t	Lκ
			[mm]	[mm]		[N]	[kN]	[kN]	[N/µm]		[mm] [mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]		[mm]	[mm]	[mm]	[mm]	[mm]
14 ⁴ 12 ⁴	6 → 2.16 3 → 2.16	6.1,5.3 6.1,5.4	2 2	16 16	3 4	1.5 1.5	2.9 3.8	4.8 6.4	200 260	1416	39 43	28 28	10 10	38 38	5.5 5.5	48 48	10 10	6 6	20 20	1213	21 25	28 28	5 5	1.9 1.9	12 16
	4.16 4.16	5.3.3 5.3.4	4 4	16 16	3 4	3 3	8.9 11.3	11.3 15.1	210 280		49 53	28 28	10 10	38 38	5.5 5.5	48 48	10 10	6 6	20 20		23 27	28 28	5 5	1.9 1.9	12 12
	5.16 5.16	5.3,5.3 5.3,5.4	5 5	16 16	3 4	3.5 3.5	10.1 12.9	11.9 15.9	180 250		54 59	28 28	10 10	38 38	5.5 5.5	48 48	10 10	6 6	20 20		29 34	28 28	5 5	1.9 1.9	16 12
242	6 10.1 10.1	16.3,5.6 16.3,5.10	10 10	16 16	6 10	3.5 3.5	19.6 31.4	27.4 47.3	270 450	2426	44 64	32 32	12 16	42 42	5.5 5.5	52 52	10 10	11 11	20 20						
14 ⁻ 12 ⁻	6→2.20 3→2.20 2.20 2.20).1,5.3).1,5.4).1,5.5	2 2 2	20 20 20	3 4 5	1.5 1.5 1.5	3.2 4.1 5	6.1 8.1 10.1	240 310 390	1416	48 52 56	36 36 36	10 10 10	47 47 47	6.6 6.6 6.6	58 58 58	10 10 10	6 6 6	22 22 22	1213	23 27 31	33 33 33	6 6 6	2.5 2.5 2.5	14 16 16
	4.20 4.20).3.3).3.4	4 4	20 20	3 4	3 3	10.1 13	14.8 19.7	270 360		49 53	36 36	10 10	47 47	6.6 6.6	58 58	10 10	6 6	22 22		23 27	33 33	6 6	2.5 2.5	14 10
	5.20 5.20).3,5.3).3,5.4	5 5	20 20	3 4	3.5 3.5	12.1 15.5	16.6 22.1	260 340		55 60	36 36	10 10	47 47	6.6 6.6	58 58	10 10	6 6	22 22		29 34	33 33	6 6	2.5 2.5	16 14
	10.2 10.2	20.3,5.2 20.3,5.3	10 10	20 20	2 3	3.5 3.5	8.5 12	10.9 16.4	130 200	1	62 76	36 36	16 16	47 47	6.6 6.6	58 58	10 10	7 7	22 22		36 48	33 33	6 6	2.5 2.5	20 25
342	6 20.2 20.2	20.3,5.4 20.3,5.6	20 20	20 20	4 6	3.5 3.5	14.7 21.7	25.3 38	140 220	3526	75 ♦ 95 ♦	36 36	16 16	47 47	6.6 6.6	58 58	10 10	7 7	22 22						

Single Nut with Ball Oversize Preload T0 - T5

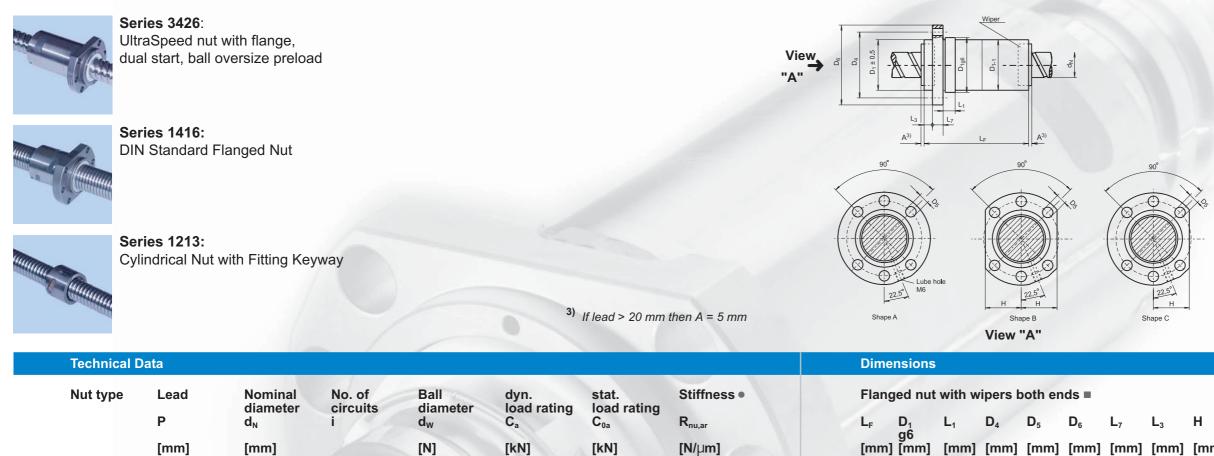






Nominal Diameter 25 mm

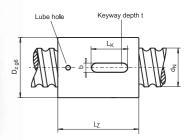
Single Nut with Ball Oversize Preload Precision Ground Execution Grade T0 - T5



	Technical Da	ata								Dime	ension	S												
	Nut type	Lead P	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating C _a	stat. load rating	Stiffness ● R _{nu,ar}		1.	ged nu	it with v	wipers D₄	/				н		with	idrical i fitting l	nut wi keywa	thout w	
		r [mm]	a _N [mm]	12	a _w [N]	(kN)	C _{0a} [kN]	[N/µm]		L _F	g6] [mm]	L ₁		D₅] [mm]	D ₆ [mm]	L ₇ [mm]	L ₃] [mm]] [mm]		Lz [mm]	g6 [[mm]	l [mm	י] [mm]	∟ _к [mm]
1416 1213	2.25.1,5.3 2.25.1,5.4 2.25.1,5.5	2 2 2	25 25 25	3 4 5	1.5 1.5 1.5	3.5 4.5 5.5	7.7 10.3 12.8	280 370 460	1416	43 51 56	40 40 40	10 10 10	51 51 51	6.6 6.6 6.6	62 62 62	10 10 10	6 6 6	24 24 24	1213	23 27 31	38 38 38	6 6 6	2.5 2.5 2.5	14 16 16
	4.25.3.3 4.25.3.4	4 4	25 25	3 4	3.0 3.0	11.4 14.6	19.1 25.5	340 450		49 53	40 40	10 10	51 51	6.6 6.6	62 62	10 10	6 6	24 24		23 27	38 38	6 6	2.5 2.5	14 10
	5.25.3,5.3 5.25.3,5.4 5.25.3,5.5	5 5 5	25 25 25	3 4 5	3.5 3.5 3.5	13.7 17.5 21.2	21.3 28.4 35.5	330 430 530		55 60 66	40 40 40	10 10 10	51 51 51	6.6 6.6 6.6	62 62 62	10 10 10	6 6 6	24 24 24		29 34 40	38 38 38	6 6 6	2.5 2.5 2.5	16 14 14
	10.25.3,5.2 10.25.3,5.3	10 10	25 25	2 3	3.5 3.5	9.6 13.6	14.1 21.2	180 270		64 78	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24		37 49	38 38	6 6	2.5 2.5	20 25
	15.25.3,5.2 15.25.3,5.3	15 15	25 25	2 3	3.5 3.5	9.4 13.4	14.0 21.0	140 200		71 92	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24		43 68	38 38	6 6	2.5 2.5	25 32
3426	20.25.3,5.4 20.25.3,5.6	20 20	25 25	4 6	3.5 3.5	17.1 25.2	33.4 50.1	230 340	3426	75 ♦ 95 ♦		16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24						
	25.25.3,5.4	25	25	4	3.5	16.7	32.8	180		81 ♦	40	16	51	6.6	62	10	7	24						

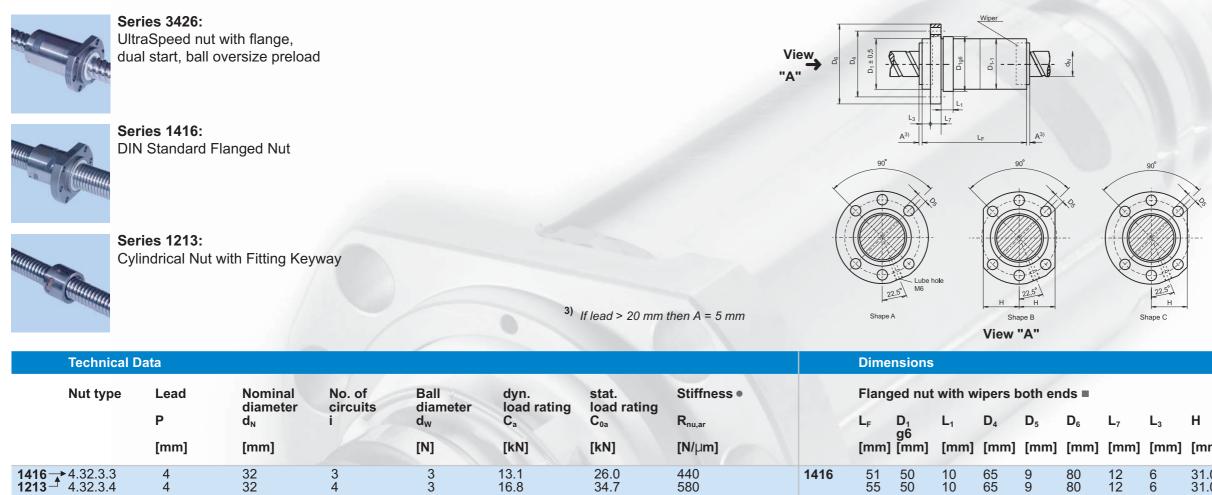
Single Nut with Ball Oversize Preload T0 - T5





Nominal Diameter 32 mm

Single Nut with Ball Oversize Preload Precision Ground Execution Grade T0 - T5



29.5

39.4 49.2

59.0

45.1

60.2

75.2

44.8

44.5

108.0

144.0

70.3

570 710

500

710

16.8

16.0

20.4

24.8

29.0

30.8

39.4

47.8

30.5

30.2

57.9

75.6

38.1

3.5 3.5 3.5 3.5

10	
reload	
Oversize P	
וBall (
with	
N	
Single	

- T5

20

5.32.3,5.3

5.32.3,5.4 5.32.3,5.5

5.32.3,5.6

10.32.6.3

10.32.6.4

10.32.6.5

15.32.6.3

20.32.6.3

20.32.6.6

20.32.6.8

30.32.6.4

32 32

32

65 65

9 9

9 9

80 80

10

16 16

62 67

95

103 56

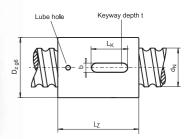
123 56

95 🔶 56

50 50

50 50





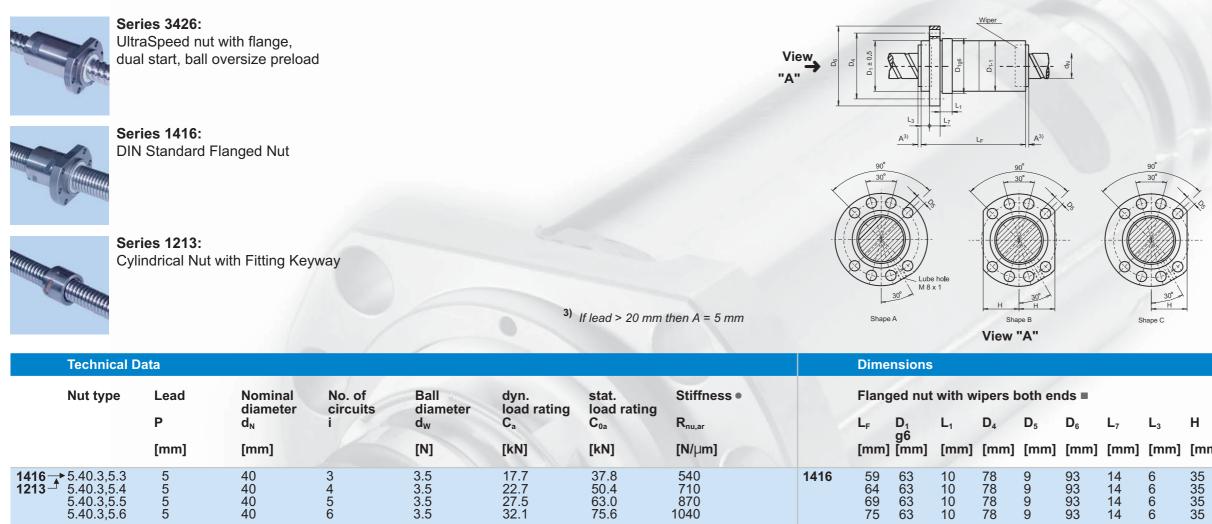
L ₃ [mm]	H [mm]		with fi L _z	drical n itting k D _z g6 [mm]	eyway b	iout wip t [mm]	pers L _κ [mm]
6 6	31.0 31.0	1213	23 27	48 48	6 6	2.5 2.5	14 16
6 6 6 6	31.0 31.0 31.0 31.0 31.0		29 34 40 48	48 48 48 48	6 6 6 6	2.5 2.5 2.5 2.5	16 20 25 32
7 7 7	31.0 31.0 31.0		56 66 77	50 50 50	8 8 8	3.1 3.1 3.1	36 32 32
7	31.0		72	50	8	3.1	45
7	31.0		95	50	8	3.1	45
7 7	32.5 32.5						
7	32.5						

12 12

12

Nominal Diameter 40 mm

Single Nut with Ball Oversize Preload Precision Ground Execution Grade T0 - T5



Technical D	ata								Dime	nsions			
Nut type	Lead	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating	stat. load rating	Stiffness •		Flang	ed nut	with w	vipers b	oth
	Ρ	d _N	i	d _w	C _a	C _{0a}	R _{nu,ar}		L _F	D₁ g6	L ₁	D ₄	D ₅
	[mm]	[mm]		[N]	[kN]	[kN]	[N/µm]		[mm]		[mm]	[mm]	[m
5.40.3,5.3 5.40.3,5.4 5.40.3,5.5 5.40.3,5.5 5.40.3,5.6	5 5 5 5 5	40 40 40 40	3 4 5 6	3.5 3.5 3.5 3.5 3.5	17.7 22.7 27.5 32.1	37.8 50.4 63.0 75.6	540 710 870 1040	1416	59 64 69 75	63 63 63 63	10 10 10 10	78 78 78 78 78	9 9 9 9
10.40.7,5.3 10.40.7,5.4 10.40.7,5.5	10 10 10	40 40 40	3 4 5	7.5 7.5 7.5	46.1 59.0 71.5	70.6 94.2 117.7	490 650 800		86 97 110	63 63 63	16 16 16	78 78 78	9 9 9
15.40.7,5.3 15.40.7,5.4	15 15	40 40	3 4	7.5 7.5	45.9 58.7	70.4 93.8	440 570		104 121	63 63	16 16	78 78	9 9
20.40.7,5.2 20.40.7,5.3	20 20	40 40	2 3	7.5 7.5	32.1 45.5	46.6 70.0	250 370		92 121	63 63	16 16	78 78	9 9
20.40.6.6 20.40.6.8	20 20	40 40	6 8	6.0 6.0	64.9 84.7	136.4 181.8	720 960	3426	105 ♦ 125 ♦		16 16	78 78	9 9
25.40.6.6 25.40.6.8	25 25	40 40	6 8	6.0 6.0	64.3 84.0	135.4 180.6	630 840	1 4	115 ♦ 135 ♦		16 16	78 78	9 9
30.40.6.6 30.40.6.8	30 30	40 40	6 8	6.0 6.0	63.6 83.1	134.3 179.1	550 730		123 ♦ 153 ♦		16 16	78 78	9 9

61.9

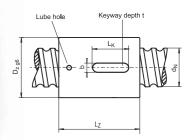
6.0

40.40.6.6

131.6

155 🔶 63





	H [mm]		with f L _z [mm]	drical n itting k D _z g6 [mm]	eyway b [mm]	t [mm]	L _K [mm]
6 6 6	35 35 35 35	1213	29 34 40 48	56 56 56 56	8 8 8 8	3.1 3.1 3.1 3.1	16 20 25 32
7 7 7	35 35 35		56 66 77	63 63 63	8 8 8	3.1 3.1 3.1	36 45 50
7 7	35 35		72 88	63 63	8 8	3.1 3.1	50 50
7 7	35 35		59 93	63 63	8 8	3.1 3.1	36 63
7 7	35 35						
7 7	35 35						
7 7	35 35						
7	35						

14

14

93 93

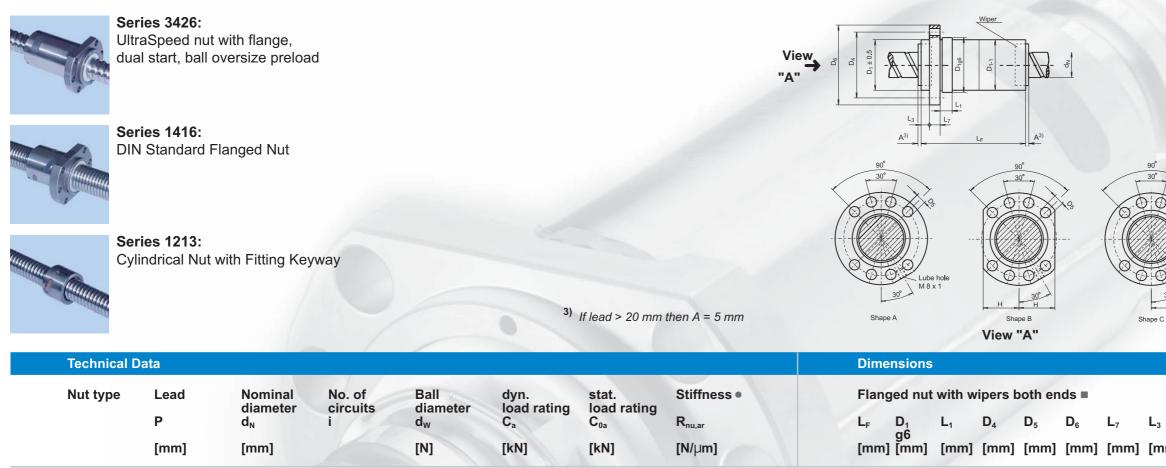
93

93

93

Nominal Diameter 50 mm

Single Nut with Ball Oversize Preload Precision Ground Execution Grade T0 - T5

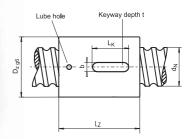


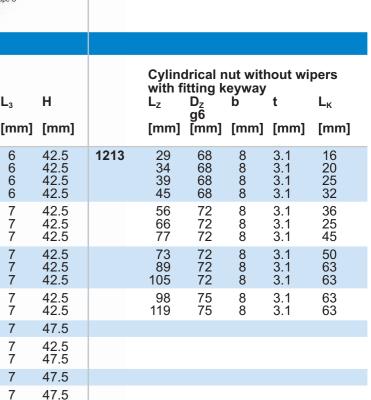
	Technical D	ata								Dimensi	ons					
	Nut type	Lead	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating	stat. load rating	Stiffness •		Flanged	nut with	wipers	both ei	nds ∎		
		Р	d _N	i	d _w	C _a	C _{0a}	R _{nu,ar}			L ₁	D_4	D ₅	D_6	L ₇	L_3
		[mm]	[mm]		[N]	[kN]	[kN]	[N /µ m]		g6 [mm] [m	m] [mm] [mm]	[mm]	[mm]	[mm]	[m
1416 1213	5.50.3,5.3 5.50.3,5.4 5.50.3,5.5 5.50.3,5.5 5.50.3,5.6	5 5 5 5 5	50 50 50 50	3 4 5 6	3.5 3.5 3.5 3.5 3.5	19.6 25.1 30.4 35.6	48.4 64.6 80.7 96.9	650 850 1060 1260	1416	66 7 71 7	5 10 5 10 5 10 5 10 5 10	93 93 93 93	11 11 11 11	110 110 110 110	16 16 16 16	6 6 6 6
	10.50.7,5.3 10.50.7,5.4 10.50.7,5.5	10 10 10	50 50 50	3 4 5	7.5 7.5 7.5	52.7 67.4 81.7	92.3 123.0 153.8	630 830 1030		88 7 99 7 111 7	5 16 5 16 5 16	93 93 93	11 11 11	110 110 110	16 16 16	7 7 7
	15.50.7,5.3 15.50.7,5.4 15.50.7,5.5	15 15 15	50 50 50	3 4 5	7.5 7.5 7.5	52.5 67.2 81.4	92.1 122.7 153.4	580 760 950		107 7 124 7 142 7	5 16 5 16 5 16	93 93 93	11 11 11	110 110 110	16 16 16	7 7 7
	20.50.9.3 20.50.9.4	20 20	50 50	3 4	9.0 9.0	76.8 98.4	128.7 171.5	590 770		129 7 150 7		93 93	11 11	110 110	16 16	7 7
3426	25.50.7,5.8	25	50	8	7.5	126.6	284.1	1190	3426	140 🔶 8	2 16	108	11	125	18	7
	30.50.6.8 30.50.7,5.8	30 30	50 50	8 8	6.0 7.5	92.6 125.7	226.6 282.6	1030 1070		160 ♦ 7 165 ♦ 8	5 16 2 16	93 108	11 11	110 125	16 18	7 7
	35.50.7,5.8	35	50	8	7.5	124.7	280.8	950		180 🔶 8	2 16	108	11	125	18	7
	40.50.7,5.6	40	50	6	7.5	94.6	209.1	640	1	160 🔶 8	2 16	108	11	125	18	7

 With standard wipers. Special execution wipers available. See page 25.

Single Nut with Ball Oversize Preload T0 - T5

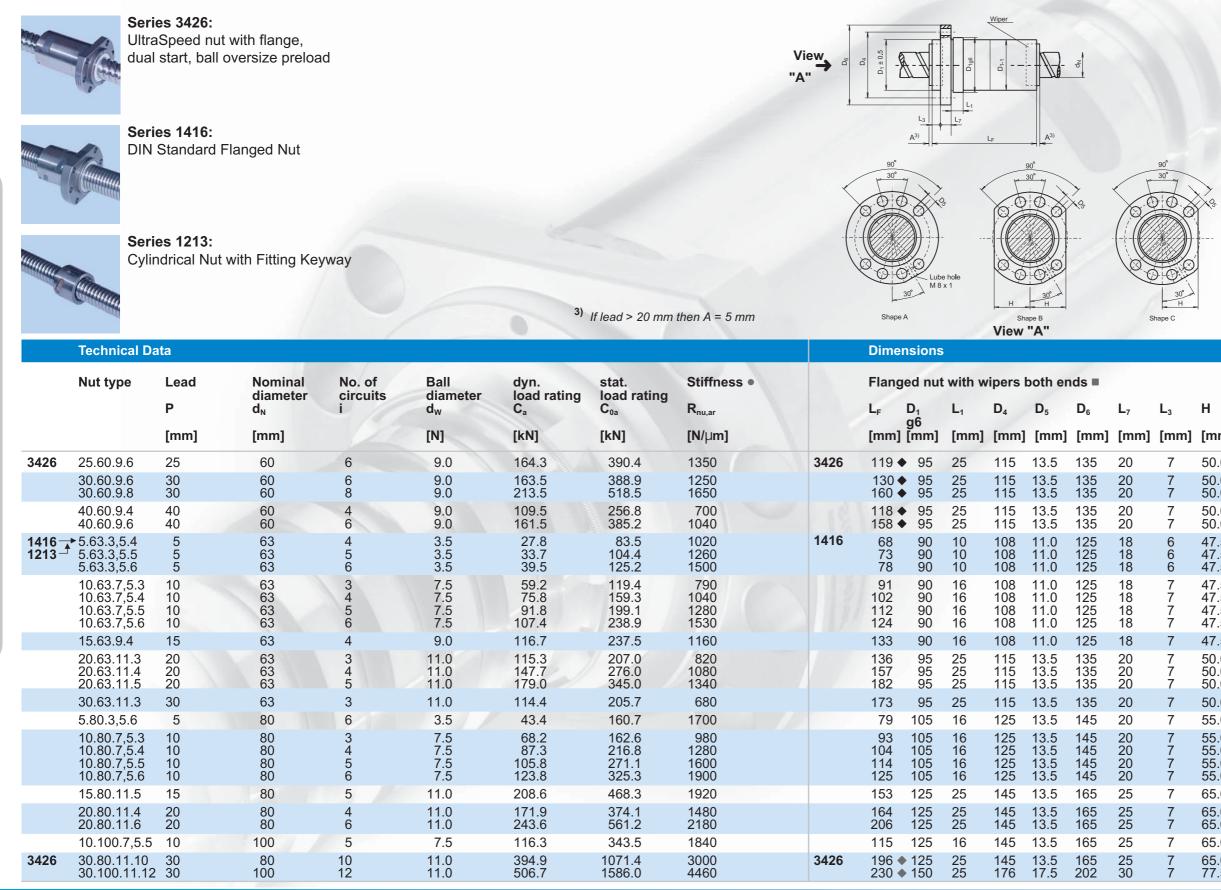






Nominal Diameter 60-100 mm

Single Nut with Ball Oversize Preload Precision Ground Execution Grade T0 - T5



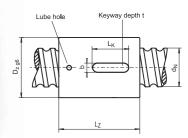
42

Actual stiffness at preload equal to 0.08 * C_a

With standard wipers.

Special execution wipers available. See page 25.





				drical n itting k		nout wij	pers
L ₃	н		Lz	D _z g6	b	t	Lκ
[mm]	[mm]		[mm]	[mm]	[mm]	[mm]	[mm]
7	50.0						
7 7	50.0 50.0						
7 7	50.0 50.0						
6 6 6	47.5 47.5 47.5	1213	34 34 45	82 82 82	8 8 8	3.1 3.1 3.1	20 20 32
7 7 7 7	47.5 47.5 47.5 47.5		56 67 78 89	85 85 85 85	8 8 8 8	3.1 3.1 3.1 3.1	36 45 50 50
7	47.5		100	88	8	3.1	50
7 7 7	50.0 50.0 50.0		98 118 151	92 92 92	8 8 8	3.1 3.1 3.1	50 50 63
7	50.0		139	92	8	3.1	50
7	55.0		44	102	8	3.1	36
7 7 7 7	55.0 55.0 55.0 55.0		56 67 77 89	102 102 102 102	8 8 8 8	3.1 3.1 3.1 3.1	36 36 45 45
7	65.0		114	102	8	3.1	45
7 7	65.0 65.0		119 170	110 110	8 8	3.1 3.1	50 50
7	65.0		76	122	8	3.1	50
7 7	65.0 77.5						

Nominal Diameter 16-20 mm

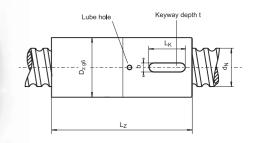
Double Nut Precision Ground Execution Grade T0 - T5



	Technical D	ata								Dimension	s												
	Nut type	Lead	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating	stat. load rating	Stiffness •		Flanged nu	t with v		_				н		with f	drical r itting k	nut wit keywa	hout wi	· .
		r [mm]	a _N [mm]	2x	a _w [N]	C _a [kN]	C _{0a} [kN]	R _{nu,ar} [N/µm]		L _F D₁ g6 [mm] [mm]	L ₁	D₄] [mm]	D₅ [mm]	D ₆ [mm]	L ₇	∟₃] [mm]			L _z [mm]	g6 [mm]	[mm]	ر [mm]	L _κ [mm]
1516 1313	► 2.16.1,5.3 2.16.1,5.4	2 2	16 16	3 4	1.5 1.5	2.9 3.8	4.8 6.4	240 330	1516	62 28 70 28	10 10	38 38	5,5 5,5	48 48	10 10	6 6	20 20	1313	42 50	28 28	5 5	1,9 1,9	12 16
	4.16.3.3 4.16.3.4	4 4	16 16	3 4	3.0 3.0	8.9 11.3	11.3 15.1	270 360		73 28 81 28	10 10	38 38	5,5 5,5	48 48	10 10	6 6	20 20		46 54	28 28	5 5	1,9 1,9	12 12
2526	5.16.3,5.3 5.16.3,5.4	5 5	16 16	3 4	3.5 3.5	10.1 12.9	11.9 15.9	240 320		84 28 95 28	10 10	38 38	5,5 5,5	48 48	10 10	6 6	20 20		58 68	28 28	5 5	1,9 1,9	16 12
	10.16.3,5.3 10.16.3,5.5	10 10	16 16	3 5	3.5 3.5	9.9 16.5	12.4 22.4	240 390	2426	44 32 64 32	12 16	42 42	5,5 5,5	52 52	10 10	6 6	20 20						
1516 1313	2.20.1,5.3 2.20.1,5.4 2.20.1,5.5	2 2 2	20 20 20	3 4 5	1.5 1.5 1.5	3.2 4.1 5.0	6.1 8.1 10.1	300 390 480	1516	72 36 80 36 89 36	10 10 10	47 47 47	6,6 6,6 6,6	58 58 58	10 10 10	6 6 6	22 22 22	1313	46 54 62	33 33 33	6 6 6	2,5 2,5 2,5	14 16 16
	4.20.3.3 4.20.3.4	4 4	20 20	3 4	3.0 3.0	10.1 13.0	14.8 19.7	350 460		73 36 82 36	10 10	47 47	6,6 6,6	58 58	10 10	6 6	22 22		46 54	33 33	6 6	2,5 2,5	14 10
	5.20.3,5.3 5.20.3,5.4	5 5	20 20	3 4	3.5 3.5	12.1 15.5	16.6 22.1	330 440		85 36 95 36	10 10	47 47	6,6 6,6	58 58	10 10	6 6	22 22		58 68	33 33	6 6	2,5 2,5	16 14
	10.20.3,5.2 10.20.3,5.3	10 10	20 20	2 3	3.5 3.5	8.5 12.0	10.9 16.4	180 270		102 36 126 36	16 16	47 47	6,6 6,6	58 58	10 10	7 7	22 22		72 96	33 33	6 6	2,5 2,5	20 25
3526	20.20.3,5.2 20.20.3,5.3	20 20	20 20	2 3	3.5 3.5	7.1 11.0	12.7 19.0	130 200	3526	75 ♦ 36 95 ♦ 36	16 16	47 47	6,6 6,6	58 58	10 10	7 7	22 22						

Double Nut T0 - T5







Nominal Diameter 25 mm

Double Nut Precision Ground Execution Grade T0 - T5



Series 3526:

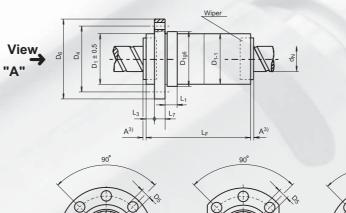
UltraSpeed nut with flange, dual start, pitch offset preload



Series 1516: DIN Standard Flanged Nut with UNILOCK preload



Series 1313:



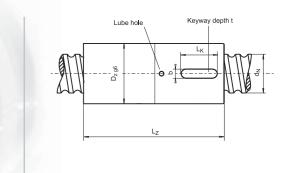
Lube hole



Cylindrical Nut with Fitting Keyway with UNILOCK preload

					3)	If lead > 20 mm	then A = 5 mm		Shape A				ape B "A"			Shape C							
Technical [Data								Dimer	nsions	s												
Nut type	Lead	Nominal	No. of	Ball	dyn.	stat.	Stiffness •		Flang	ed nu	t with v	vipers I	both e	nds ∎					Cylin	drical	nut wi	ithout wi	ipers
	Р	diameter d _N	circuits i	diameter d _w	load rating C _a	load rating C _{0a}	R _{nu,ar}		LF		L ₁	D_4	D ₅	D_6	L ₇	L ₃	н		L_z	fitting k D _z	b	y t	Lκ
	[mm]	[mm]	2x	[N]	[kN]	[kN]	[N/µm]		[mm]	go [mm]	[mm]	[mm]	[mm]	[mm]	[[mm]] [mm]	[[mm]		[mm]	90 [mm]	[mm] [mm]	[mm]
1516 2.25.1,5.3 1313 2.25.1,5.4 2.25.1,5.5	2 2 2	25 25 25	3 4 5	1.5 1.5 1.5	3.5 4.5 5.5	7.7 10.3 12.8	350 460 570	1516	72 77 81	40 40 40	10 10 10	51 51 51	6.6 6.6 6.6	62 62 62	10 10 10	6 6 6	24 24 24	1313	46 54 62	38 38 38	6 6 6	2.5 2.5 2.5	14 16 16
4.25.3.3 4.25.3.4	4 4	25 25	3 4	3.0 3.0	11.4 14.6	19.1 25.5	430 570		73 82	40 40	10 10	51 51	6.6 6.6	62 62	10 10	6 6	24 24		46 54	38 38	6 6	2.5 2.5	14 10
5.25.3,5.3 5.25.3,5.4 5.25.3,5.5	5 5 5	25 25 25	3 4 5	3.5 3.5 3.5	13.7 17.5 21.2	21.3 28.4 35.5	410 550 680		85 95 107	40 40 40	10 10 10	51 51 51	6.6 6.6 6.6	62 62 62	10 10 10	6 6 6	24 24 24		58 68 80	38 38 38	6 6 6	2.5 2.5 2.5	16 14 14
10.25.3,5.2 10.25.3,5.3	10 10	25 25	23	3.5 3.5	9.6 13.6	14.1 21.2	240 350		104 128	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24		74 98	38 38	6 6	2.5 2.5	20 25
15.25.3,5.2 15.25.3,5.3	15 15	25 25	2 3	3.5 3.5	9.4 13.4	14.0 21.0	190 290		117 167	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24		86 136	38 38	6 6	2.5 2.5	25 32
3526 20.25.3,5.2 20.25.3,5.3	20 20	25 25	2 3	3.5 3.5	8.2 12.8	16.7 25.0	200 300	3526	75 ♦ 95 ♦		16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24						
25.25.3,5.2		25	2	3.5	8.1	16.4	160		81 ♦		16	51	6.6	62	10	7	24						







Nominal Diameter 32 mm

Double Precision Ground Execution Grade T0 - T5



Series 3526:

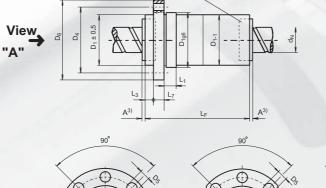
UltraSpeed nut with flange, dual start, pitch offset preload



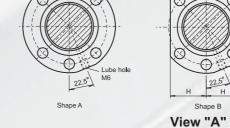
Series 1516: DIN Standard Flanged Nut with UNILOCK preload



Series 1313: Cylindrical Nut with Fitting Keyway with UNILOCK preload



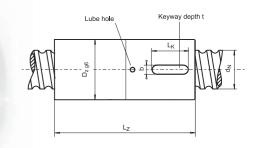
Shape B



	Technical D)ata								Dimer	nsions	S												
	Nut type	Lead P	Nominal diameter d _N	No. of circuits i	Ball diameter d _w	dyn. Ioad rating C _a	stat. Ioad rating C _{0a}	Stiffness ● R _{nu,ar}		Flang L _F	ed nu D ₁	t with v	wipers D₄	both e D₅	nds ∎ D₀	L ₇	L ₃	н		Cylin with f L _z	drical itting D _z	nut wi keywa b	thout w y t	ipers L _κ
		[mm]	[mm]	2x	[N]	[kN]	[kN]	[N /µm]		[mm]	go [mm]	[mm]] [mm]	[mm] [mm]	[mm]	[mm]	[mm]		[mm]	go [mm]	[mm] [mm]	[mm]
1516 1313	► 4.32.3.3 4.32.3.4	4 4	32 32	3 4	3.0 3.0	13.1 16.8	26.0 34.7	550 730	1516	75 84	50 50	10 10	65 65	9 9	80 80	12 12	6 6	31.0 31.0	1313	46 54	48 48	6 6	2.5 2.5	14 16
	5.32.3,5.3 5.32.3,5.4 5.32.3,5.5 5.32.3,5.6	5 5 5 5	32 32 32 32 32	3 4 5 6	3.5 3.5 3.5 3.5 3.5	16.0 20.4 24.8 29.0	29.5 39.4 49.2 59.0	550 720 890 1060		87 97 107 114	50 50 50 50	10 10 10 10	65 65 65 65	9 9 9 9	80 80 80 80	12 12 12 12	6 6 6	31.0 31.0 31.0 31.0 31.0		58 68 80 96	48 48 48 48	6 6 6	2.5 2.5 2.5 2.5	16 20 25 32
	10.32.6.3 10.32.6.4 10.32.6.5	10 10 10	32 32 32	3 4 5	6.0 6.0 6.0	30.8 39.4 47.8	45.1 60.2 75.2	490 640 750		144 165 187	50 50 50	16 16 16	65 65 65	9 9 9	80 80 80	12 12 12	7 7 7	31.0 31.0 31.0		112 132 154	50 50 50	8 8 8	3.1 3.1 3.1	36 32 32
	15.32.6.3	15	32	3	6.0	30.5	44.8	420		177	50	16	65	9	80	12	7	31.0		144	50	8	3.1	45
	20.32.6.3	20	32	3	6.0	30.2	44.5	350		200	50	16	65	9	80	12	7	31.0		190	50	8	3.1	45
3526	20.32.6.3 20.32.6.4	20 20	32 32	3 4	6.0 6.0	29.4 39.3	54.0 72.0	460 610	3526	103 • 123 •		16 16	71 71	9 9	86 86	12 12	7 7	32.5 32.5						
	30.32.6.2	30	32	2	6.0	18.4	35.2	220		95 🔷	56	16	71	9	86	12	7	32.5						

³⁾ If lead > 20 mm then A = 5 mm







Nominal Diameter 40 mm

Double Nut Precision Ground Execution Grade T0 - T5



Series 3526: UltraSpeed nut with flange,

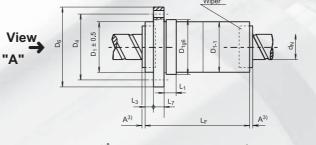
dual start, pitch offset preload

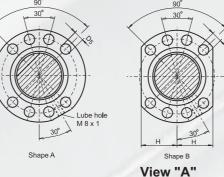


Series 1516: DIN Standard Flanged Nut with UNILOCK preload



Series 1313: Cylindrical Nut with Fitting Keyway with UNILOCK preload





OC

	Technical D	ata								Dimen	isions	;												
	Nut type	Lead P	Nominal diameter d _N	No. of circuits i	Ball diameter d _w	dyn. Ioad rating C _a	stat. Ioad rating C _{0a}	Stiffness ● R _{nu,ar}		L _F	D₁ g6	t with w L ₁	D ₄	D ₅	D ₆	L ₇	L ₃	н		with L _z	fitting D _z g6	keywa b	t	L _K
		[mm]	[mm]	2x	[N]	[kN]	[kN]	[N/µm]		[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm	[[mm]] [mm]		[mm]	[mm]	[mm] [mm]	[mm]
1516 1313	► 5.40.3,5.3 5.40.3,5.4 5.40.3,5.5 5.40.3,5.6	5 5 5 5 5	40 40 40 40	3 4 5 6	3.5 3.5 3.5 3.5 3.5	17.7 22.7 27.5 32.1	37.8 50.4 63.0 75.6	670 880 1100 1300	1516	89 99 109 121	63 63 63 63	10 10 10 10	78 78 78 78	9 9 9 9	93 93 93 93	14 14 14 14	6 6 6	35.0 35.0 35.0 35.0	1313	58 68 80 96	56 56 56 56	8 8 8	3.1 3.1 3.1 3.1	16 20 25 32
	10.40.7,5.3 10.40.7,5.4	10 10	40 40	3 4	7.5 7.5	46.1 59.0	70.6 94.2	630 830		147 168	63 63	16 16	78 78	9 9	93 93	14 14	7 7	35.0 35.0		112 132	63 63	8 8	3.1 3.1	36 45
	10.40.7,5.5	10	40	5	7.5	71.5	117.7	1030		189	63	16	78	9	93	14	7	35.0		154	63	8	3.1	50
	15.40.7,5.3 15.40.7,5.4	15 15	40 40	3 4	7.5 7.5	45.9 58.7	70.4 93.8	570 750		180 211	63 63	16 16	78 78	9 9	93 93	14 14	7 7	35.0 35.0		144 176	63 63	8 8	3.1 3.1	50 50
	20.40.7,5.2 20.40.7,5.3	20 20	40 40	2 3	7.5 7.5	32.1 45.5	46.6 70.0	340 500		153 221	63 63	16 16	78 78	9 9	93 93	14 14	7 7	35.0 35.0		118 186	63 63	8 8	3.1 3.1	36 63
3526	20.40.6.3 20.40.6.4 20.40.6.5	20 20 20	40 40 40	3 4 5	6.0 6.0 6.0	32.9 44.0 54.6	68.2 90.9 113.6	610 800 1000	3526	105 ♦ 125 ♦ 145 ♦	63	16 16 16	78 78 78	9 9 9	93 93 93	14 14 14	7 7 7	35.0 35.0 35.0						
	25.40.6.3 25.40.6.4	25 25	40 40	3 4	6.0 6.0	32.6 43.6	67.7 90.3	540 720		115 ♦ 135 ♦	63 63	16 16	78 78	9 9	93 93	14 14	7 7	35.0 35.0						
	30.40.6.3 30.40.6.4	30 30	40 40	3 4	6.0 6.0	32.3 43.1	67.2 89.5	490 650	1	123 ♦ 153 ♦	63 63	16 16	78 78	9 9	93 93	14 14	7 7	35.0 35.0						
	40.40.6.3	40	40	3	6.0	31.4	65.8	390		155 🔶	63	16	78	9	93	14	7	35.0						

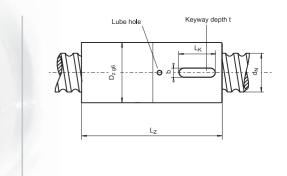
³⁾ If lead > 20 mm then A = 5 mm

Precision Ballscrews

 With standard wipers. Special execution wipers available. See page 25.

50





Nominal Diameter 50 mm

Double Nut Precision Ground Execution Grade T0 - T5



Series 3526:

UltraSpeed nut with flange, dual start, pitch offset preload

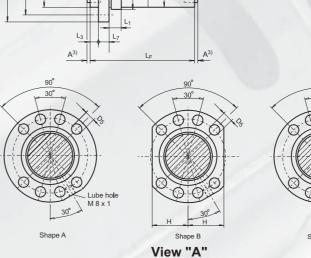


Series 1516: DIN Standard Flanged Nut with UNILOCK preload



Series 1313: Cylindrical Nut with Fitting Keyway with UNILOCK preload

³⁾ If lead > 20 mm then A = 5 mm

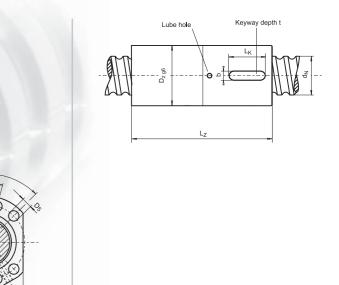


	Technical Da	ata								Dimen	sions	S												
	Nut type	Lead	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating	stat. load rating	Stiffness •		1		t with v		1				н		with f	drical ı itting k	nut wi keywa	hout w	· .
		r [mm]	a _N [mm]	2x	d _w [N]	Ca [kN]	C _{0a} [kN]	R _{nu,ar} [N/µm]		L _F	D₁ g6 [mm]	L ₁ [mm]	D₄ [mm]	D₅ [mm	D ₆] [mm]	L ₇ [mm]	L ₃			Lz [mm]	g6 [mm]	D [mm	י [mm]	∟ _κ [mm]
1516 1313	5.50.3,5.3 5.50.3,5.4 5.50.3,5.5 5.50.3,5.5 5.50.3,5.6	5 5 5 5	50 50 50 50	3 4 5 6	3.5 3.5 3.5 3.5 3.5	19.6 25.1 30.4 35.6	48.4 64.6 80.7 96.9	800 1060 1320 1570	1516	91 101 111 122	75 75 75 75	10 10 10 10	93 93 93 93	11 11 11 11	110 110 110 110	16 16 16 16	6 6 6	42.5 42.5 42.5 42.5	1313	58 68 78 90	68 68 68 68	8 8 8	3.1 3.1 3.1 3.1	1 20 25 32
	10.50.7,5.3 10.50.7,5.4 10.50.7,5.5	10 10 10	50 50 50	3 4 5	7.5 7.5 7.5	52.7 67.4 81.7	92.3 123.0 153.8	800 1060 1310		148 169 191	75 75 75	16 16 16	93 93 93	11 11 11	110 110 110	16 16 16	7 7 7	42.5 42.5 42.5		112 132 154	72 72 72	8 8 8	3.1 3.1 3.1	36 25 45
	15.50.7,5.3 15.50.7,5.4	15 15	50 50	3 4	7.5 7.5	52.5 67.2	92.1 122.7	760 990		182 213	75 75	16 16	93 93	11 11	110 110	16 16	7 7	42.5 42.5		146 178	72 72	8 8	3.1 3.1	50 63
	15.50.7,5.5 20.50.9.3	15 20	50 50	5 3	7.5 9.0	81.4 76.8	153.4 128.7	1230 780		262 229	75 75	16 16	93 93	11 11	110 110	16 16	7 7	42.5 42.5		210 196	72 75	8 8	3.1 3.1	63 63
	20.50.9.4	20	50	4	9.0	98.4	171.5	1020	11	271	75	16	93	11	110	16	7	42.5		238	75	8	3.1	63
3526	25.50.7,5.4	25	50	4	7.5	65.7	142.1	1000	3526	140 🔶	82	16	108	11	125	18	7	47.5						
	30.50.6.4 30.50.7,5.4	30 30	50 50	4 4	6.0 7.5	48.1 65.3	113.3 141.3	880 920		160 ♦ 165 ♦		16 16	93 108	11 11	110 125	16 18	7 7	42.5 47.5						
	35.50.7,5.4	35	50	4	7.5	64.7	140.4	840		180 🔶	82	16	108	11	125	18	7	47.5						
	40.50.7,5.3	40	50	3	7.5	48.0	104.6	570		160 🔶	82	16	108	11	125	18	7	47.5						

View

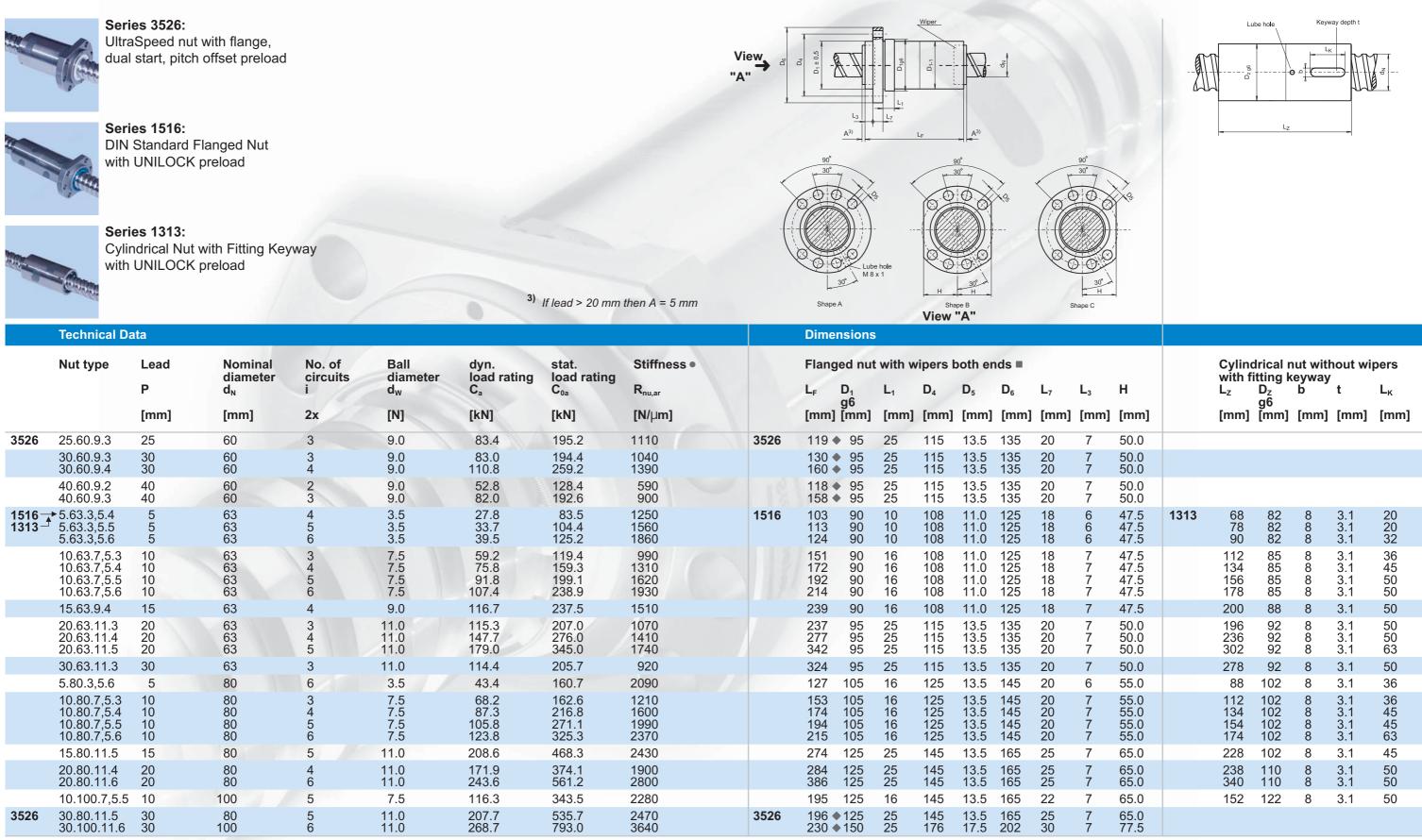
Precision Ballscrews 52





Nominal Diameter 60-100 mm

Double Nut Precision Ground Execution Grade T0 - T5



Precision Ballscrews

Actual stiffness at preload equal to 0.01 * C_a

With standard wipers.

Special execution wipers available. See page 25.

- T5

Double Nut T0



			with fi	itting k	evwav	iout wi	pers
L ₃	Н		Lz	D _z g6	b	t	L_{κ}
[mm]	[mm]		[mm]		[mm]	[mm]	[mm]
7	50.0						
7 7	50.0 50.0						
7 7	50.0 50.0						
6 6 6	47.5 47.5 47.5	1313	68 78 90	82 82 82	8 8 8	3.1 3.1 3.1	20 20 32
7 7 7 7	47.5 47.5 47.5 47.5		112 134 156 178	85 85 85 85	8 8 8	3.1 3.1 3.1 3.1	36 45 50 50
7	47.5		200	88	8	3.1	50
7 7 7	50.0 50.0 50.0		196 236 302	92 92 92	8 8 8	3.1 3.1 3.1	50 50 63
7	50.0		278	92	8	3.1	50
6	55.0		88	102	8	3.1	36
7 7 7 7	55.0 55.0 55.0 55.0		112 134 154 174	102 102 102 102	8 8 8 8	3.1 3.1 3.1 3.1	36 45 45 63
7	65.0		228	102	8	3.1	45
7 7	65.0 65.0		238 340	110 110	8 8	3.1 3.1	50 50
7	65.0		152	122	8	3.1	50
7 7	65.0 77.5						

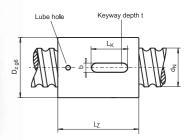
Nominal Diameter 16-20 mm

Single Nut with Backlash Ball Oversize Preload Optional Precision Rolled Execution Grade T7 - T10



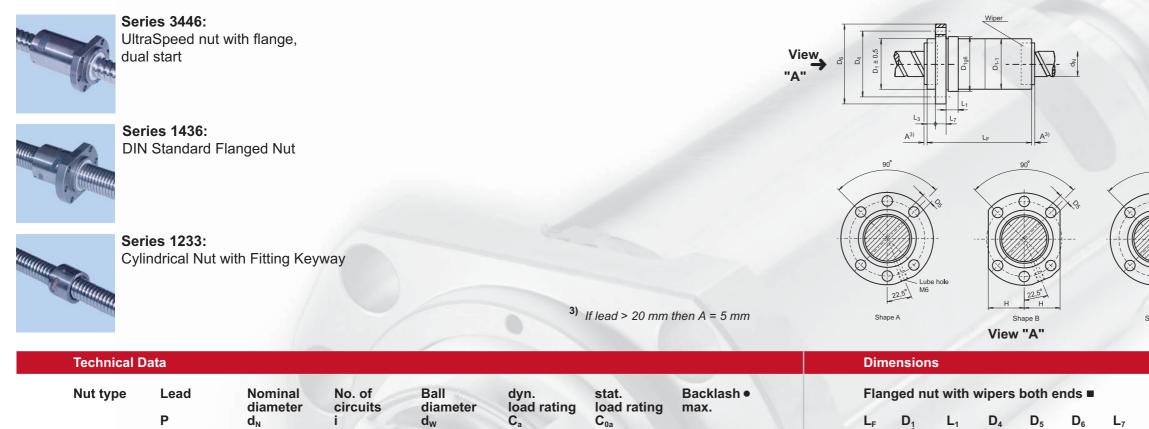
	Technical Da	ita								Dime	ension	S												
	Nut type	Lead	Nominal diameter	No. of circuits	Ball diameter	dyn. Ioad rating	stat. load rating	Backlash ● max.		Flang	ged nu	it with v	vipers	both er	nds ∎						drical ı itting k		hout w	ipers
		Ρ	d _N	i	d _w	C _a	C _{0a}	max.		L _F		L ₁	D ₄	D5	D_6	L ₇	L_3	н		Lz	D _z	b	ť	Lκ
		[mm]	[mm]		[N]	[kN]	[kN]	[mm]		[mm]] [mm]	[mm]	[mm]	[mm]	[mm]	[mm]] [mm]	[mm]		[mm]	[mm]	[mm]	[mm]	[mm]
1436 1233	► 2.16.1,5.3 2.16.1,5.4	2 2	16 16	3 4	1.5 1.5	2.9 3.8	4.8 6.4	0.02 0.02	1436	45 49	28 28	10 10	38 38	5.5 5.5	48 48	10 10	6 6	20 20	1233	21 25	28 28	5 5	1.9 1.9	12 16
	4.16.3.3 4.16.3.4	4 4	16 16	3 4	3.0 3.0	6.9 8.8	8.3 11.1	0.03 0.03		49 53	28 28	10 10	38 38	5.5 5.5	48 48	10 10	6 6	20 20		23 27	28 28	5 5	1.9 1.9	12 12
	5.16.3,5.3 5.16.3,5.4	5 5	16 16	3 4	3.5 3.5	7.9 10.1	8.8 11.7	0.04 0.04		54 59	28 28	10 10	38 38	5.5 5.5	48 48	10 10	6 6	20 20		29 34	28 28	5 5	1.9 1.9	16 12
2446	10.16.3,5.6 10.16.3,5.10	10 10	16 16	6 10	3.5 3.5	15.3 24.5	20.2 34.9	0.04 0.04	2446	44 64	32 32	12 16	42 42	5.5 5.5	52 52	10 10	11 11	20 20						
1436	2.20.1,5.3 2.20.1,5.4 2.20.1,5.5	2 2 2	20 20 20	3 4 5	1.5 1.5 1.5	3.2 4.1 5.0	6.1 8.1 10.1	0.02 0.02 0.02	1436	48 52 56	36 36 36	10 10 10	47 47 47	6.6 6.6 6.6	58 58 58	10 10 10	6 6 6	22 22 22	1233	23 27 31	33 33 33	6 6 6	2.5 2.5 2.5	14 16 16
	4.20.3.3 4.20.3.4	4 4	20 20	3 4	3.0 3.0	7.9 10.1	10.9 14.5	0.03 0.03		49 53	36 36	10 10	47 47	6.6 6.6	58 58	10 10	6 6	22 22		23 27	33 33	6 6	2.5 2.5	14 10
	5.20.3,5.3 5.20.3,5.4	5 5	20 20	3 4	3.5 3.5	9.5 12.1	12.2 16.3	0.04 0.04		55 60	36 36	10 10	47 47	6.6 6.6	58 58	10 10	6 6	22 22		29 34	33 33	6 6	2.5 2.5	16 14
	10.20.3,5.2 10.20.3,5.3	10 10	20 20	2 3	3.5 3.5	6.6 9.3	8.1 12.1	0.04 0.04	1	62 76	36 36	16 16	47 47	6.6 6.6	58 58	10 10	7 7	22 22		36 48	33 33	6 6	2.5 2.5	20 25
3446	20.20.3,5.4 20.20.3,5.6	20 20	20 20	4 6	3.5 3.5	11.5 17.0	18.7 28.1	0.04 0.04	3446	75 ♦ 95 ♦	36 36	16 16	47 47	6.6 6.6	58 58	10 10	7 7	22 22						





Nominal Diameter 25 - 32 mm

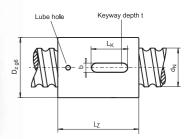
Single Nut with Backlash Ball Oversize Preload Optional Precision Rolled Execution Grade T7 - T10



	Technical D	ata								Dime	ensions													
	Nut type	Lead P	Nominal diameter d _N	No. of circuits i	Ball diameter d _w	dyn. Ioad rating C _a	stat. Ioad rating C₀ _a	Backlash ● max.		Flanç L _F	ged nut D₁ g6	with w	vipers b D ₄	ooth er D₅	nds∎ D₀	L ₇	L ₃	н			drical fitting I D _z g6		thout w y t	ipers L _κ
		[mm]	[mm]		[N]	[kN]	[kN]	[mm]		[mm]] [mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[[mm]	[mm]		[mm]	[mm]	[mm] [mm]	[mm]
143 123	6 → 2.25.1,5.3 3 2.25.1,5.4 2.25.1,5.5	2 2 2	25 25 25	3 4 5	1.5 1.5 1.5	3.5 4.5 5.5	7.7 10.3 12.8	0.02 0.02 0.02	1436	48 51 56	40 40 40	10 10 10	51 51 51	6.6 6.6 6.6	62 62 62	10 10 10	6 6 6	24 24 24	1233	23 27 31	38 38 38	6 6 6	2.5 2.5 2.5	14 16 16
	4.25.3.3 4.25.3.4	4 4	25 25	3 4	3.0 3.0	8.9 11.4	14.1 18.8	0.03 0.03		49 53	40 40	10 10	51 51	6.6 6.6	62 62	10 10	6 6	24 24		23 27	38 38	6 6	2.5 2.5	14 10
	5.25.3,5.3 5.25.3,5.4 5.25.3,5.5	5 5 5	25 25 25	3 4 5	3.5 3.5 3.5	10.7 13.6 16.5	15.7 21.0 26.2	0.04 0.04 0.04		55 60 66	40 40 40	10 10 10	51 51 51	6.6 6.6 6.6	62 62 62	10 10 10	6 6 6	24 24 24		29 34 40	38 38 38	6 6 6	2.5 2.5 2.5	16 14 14
	10.25.3,5.2 10.25.3,5.3	10 10	25 25	23	3.5 3.5	7.5 10.6	10.4 15.6	0.04 0.04		64 78	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24		37 49	38 38	6 6	2.5 2.5	20 25
	15.25.3,5.2 15.25.3,5.3	15 15	25 25	2 3	3.5 3.5	7.4 10.4	10.3 15.5	0.04 0.04		71 92	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24		43 68	38 38	6 6	2.5 2.5	25 32
344	6 20.25.3,5.4 20.25.3,5.6	20 20	25 25	4 6	3.5 3.5	13.3 19.6	24.7 37.0	0.04 0.04	3446	75 ♦ 95 ♦	40 40	16 16	51 51	6.6 6.6	62 62	10 10	7 7	24 24						
	25.25.3,5.4	25	25	4	3.5	13.0	24.2	0.04		81 🔶	40	16	51	6.6	62	10	7	24						
143 123	6 →4.32.3.3 3 →4.32.3.4	4 4	32 32	3 4	3.0 3.0	10.2 13.1	19.2 25.6	0.03 0.03	1436	51 55	50 50	10 10	65 65	9.0 9.0	80 80	12 12	6 6	31 31	1233	23 27	48 48	6 6	2.5 2.5	14 16
	5.32.3,5.3 5.32.3,5.4 5.32.3,5.5 5.32.3,5.6	5 5 5 5 5	32 32 32 32 32	3 4 5 6	3.5 3.5 3.5 3.5 3.5	12.5 15.9 19.3 22.6	21.8 29.1 36.3 43.6	0.04 0.04 0.04 0.04		57 62 67 73	50 50 50 50	10 10 10 10	65 65 65 65	9.0 9.0 9.0 9.0	80 80 80 80	12 12 12 12	6 6 6	31 31 31 31		29 34 40 48	48 48 48 48	6 6 6 6	2.5 2.5 2.5 2.5	16 20 25 32

Single Nut with Backlash T7 - T10







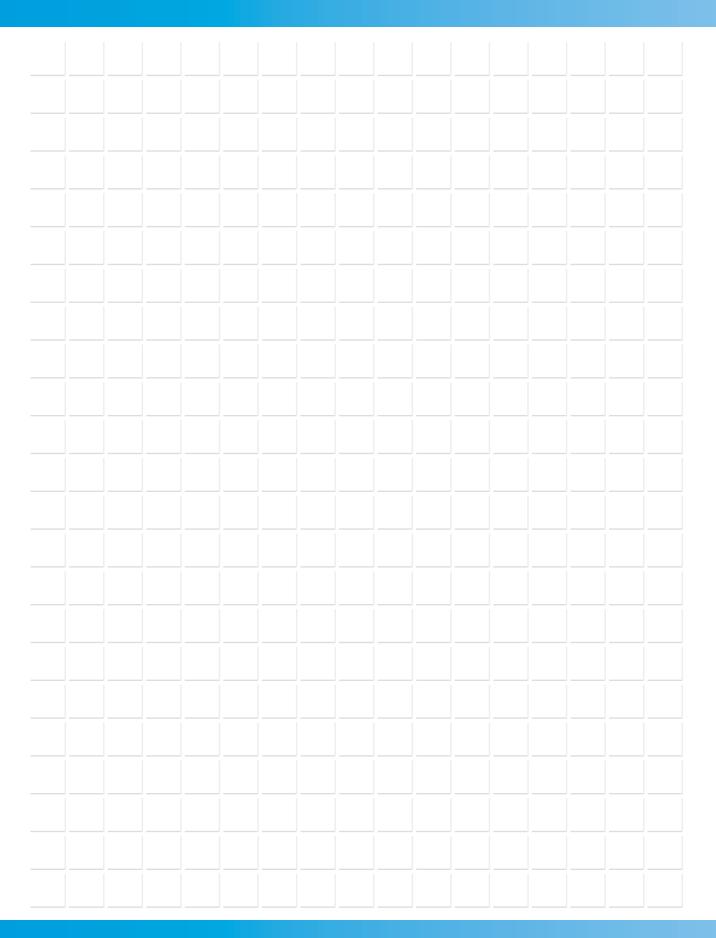
Steinmeyer Precision Ballscrews worldwide

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USA

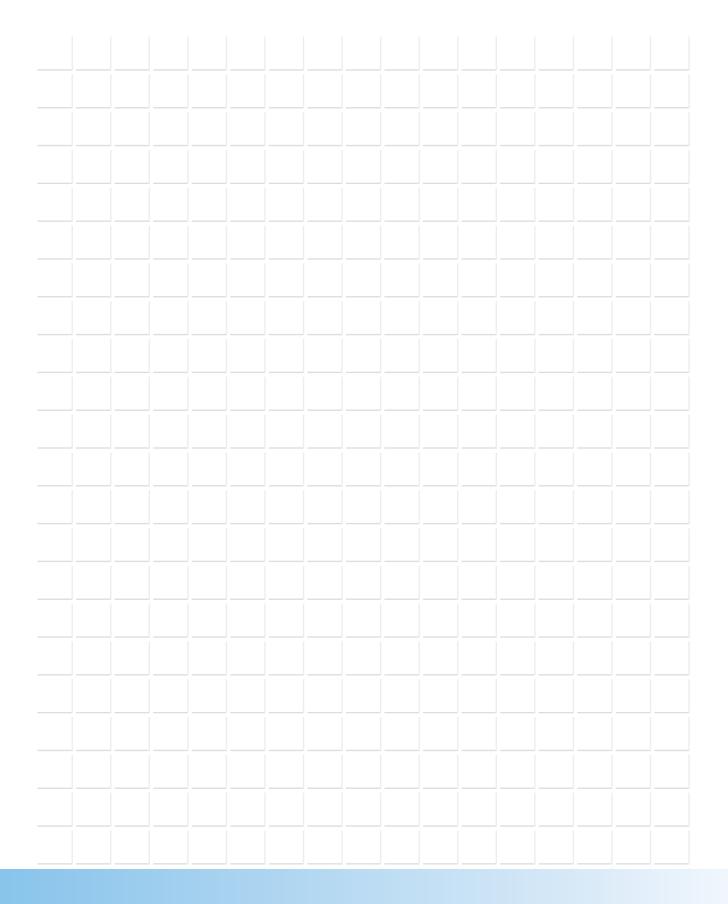
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