



Series BSB..-SU

SCHAEFFLER GROUP

added competence

The right product for every application

Optimum benefit for you With their forward-looking bearing arrangement solutions for feed spindles, main spindles, rotary tables and linear guidance units, INA and FAG have been at the forefront of the world market for decades. The bearing components alone, however, are often no longer the decisive factor for these machine subsystems.

Our customers have of course benefitted directly from significant performance improvements and unique selling propositions thanks to our "ready-to-fit" products; these compact, ready-to-fit bearings are used in accordance with the simple principle: unpack, screw mount, use. In order to optimise the entire machine tool system, however, it is becoming ever more important not simply to support the subsystems but to integrate important functions such as measurement, sealing, lubrication, braking etc. in the components themselves. This intellectual approach is fulfilled comprehensively by the new concept **added competence** in the Production Machinery Sector since it attaches central importance to systems solution thinking for the bearing, bearing position and entire system. This means that you can now access a product range that gives optimum coverage to all your applications in the machine tool.

Since direct drives and mechatronic solutions are used ever more frequently in machine tools, we have incorporated a further strong partner in the form of IDAM – INA Drives & Mechatronics – in our spectrum of capabilities. In this way, we can now supply you from a single source with bearing elements and the appropriate drive system to give complete systems that are precisely matched to each other. This opens up completely new technical and economic design possibilities for your requirements as well as significant advantages in the time and process chain.

In terms of products, we offer you a comprehensive, precisely balanced range, precision technology and top product quality. In order to match the pulse of your developments as closely as possible, furthermore, we have a worldwide network of engineers, service and sales technicians working for you and ensuring that we maintain close contact with you in your own location.

In conclusion, we are convinced that we will always have the right product for your application. Just contact us to see what we can do for you.

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Features

Bearings of series BSB..-SU are single row axial angular contact ball bearings. Due to the 60° contact angle, they can support high axial forces from one side.

Variable set combinations

The inner ring, rolling element set and outer ring are matched to each other such that a wide range of set combinations can be achieved simply using a single bearing type, *Figure 1*.

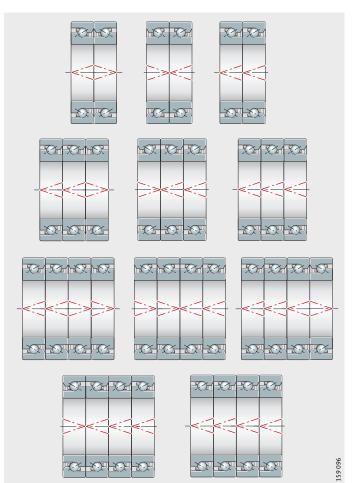


Figure 1 Set combinations

Marking of the mounting position The bearing

The bearings have marking by means of arrows on the outside diameter to indicate the contact angle arrangement, *Figure 1*:

- <> is an 0 arrangement
- >< is an X arrangement</p>
- << is a tandem arrangement.</p>

In bearings with seals (BSB..-2Z-SU), the contact angle can also be recognised from the seal colour after mounting:

- A brown seal on the outside indicates an O arrangement (side with high inner ring shoulder)
- A red seal on the outside indicates an X arrangement (side with low inner ring shoulder).

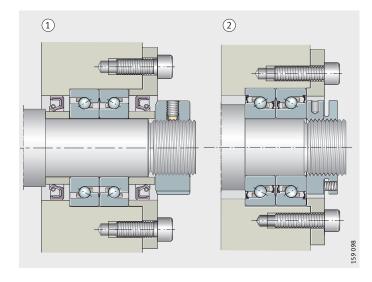
Three versions

The bearings are available in the versions BSB..-2Z-SU, BSB..-SU und BSB..-SU-L055.

With minimal gap seals, greased – BSB.-2Z-SU Due to the non-contact minimal gap seals integrated in the standard design envelope and greasing with a lithium soap grease to GA28, these bearings are maintenance-free and are thus particularly economical (for information on GA28, see TPI 123). There is therefore no need for lubricant feeds, separate seal contact rings or seals. This reduces the number of components, the mounting requirements, the costs of producing adjacent components, the design envelope required and the risk of errors in mounting work, *Figure 2*.

The gap seals give low bearing friction and allow high speeds with little heat generation in the bearing.

Due to the high quality FPM material, the seals are extremely resistant to aggressive media – such as oils and cooling lubricants – and thus increase the long term operational reliability of the bearing arrangement.



 Bearing arrangement sealed by rotary shaft seals
 Bearing sealed by integrated gap seals

Figure 2

Comparison of design envelope with/without integrated seals

Not sealed – BSB..-SU

Not sealed, greased – BSB..-SU-L055

Operating temperature

These bearings are the basic version, without greasing or seals. They are preferably used in existing constructions that are designed for oil lubrication.

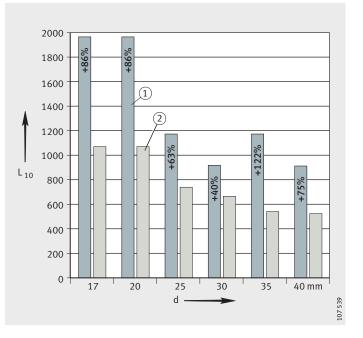
This series has no seals but is greased with a lithium soap grease to GA28 (for information on GA28, see TPI 123).

Sealed axial angular contact ball bearings are suitable for operating temperatures from -30 °C to +120 °C; this is limited by the grease, seal material and the plastic cages in the ball and cage assemblies.

Caution! The operating temperature influences the dynamic bearing characteristics. The values given in the dimension table are based on a room temperature of +20 °C.

Design and safety guidelines Load carrying capacity and rating life

Bearings of series BSB..-SU are optimised in relation to axial load carrying capacity and life. Since maximum use is made of the design envelope, high axial load carrying capacity and rigidity is achieved. Under comparable loads, this gives longer bearing arrangement life compared to bearings generally available on the market, *Figure 3*.



d = bore diameter of inner ring
① Bearing arrangement with BSB..-SU
② Competitor bearing arrangement

Figure 3

Life comparison of BSB..-SU/ competitor bearing arrangement

Life calculation

Resultant and equivalent bearing load P

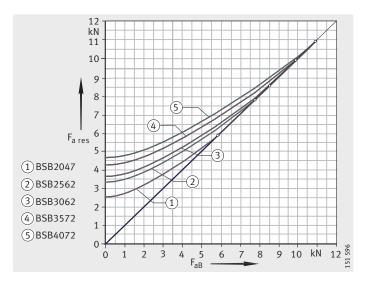
Caution!

case.

The formulae for calculating the basic rating life L_h are given in TPI 123. Calculation of the resultant and equivalent bearing load required for life calculation is described in the following paragraph and *Figure 4*.

The bearings are axially preloaded to a defined value if the recommended precision locknuts are used and the correct nut tightening torque is applied. The resultant axial bearing load $F_{a res}$ is determined from the axial operating load F_{aB} and taking account of the axial preload according to *Figure 4*, page 7. The diagrams are based on two-bearing sets in an O or X arrangement, *Figure 1*, page 4. For calculation of other set combinations, please contact us.

A load in excess of the limit value will lead to the rolling element row without load lifting off the raceway; for limit values see *Figure 4*, page 7. As a result, higher wear will occur under rapid acceleration.
For extreme moment loads and statically overdefined systems (locating/locating bearing arrangements), please contact us. The calculation program BEARINX[®] can give a precise design in this



 F_{aB} = operating load $F_{a res}$ = resultant bearing load ° = limit value

Figure 4 Resultant bearing load with BSB..-SU

Static load safety factor

The static load safety factor S_0 indicates the security against permissible permanent deformations in the bearing. Calculation of the static load safety factor is described in TPI 123. In machine tools, S_0 should be > 4.

Speeds The thermal reference speed is the speed at which an equilibrium temperature of +70 °C is calculated to be reached at a room temperature of +20 °C.

In machine tools, it is normal practice to aim for a maximum permissible bearing temperature of +50 °C. This is necessary in order to minimise the influence of heat generation on machine accuracy. The dimension tables therefore indicate practically relevant limiting speeds for machine tool applications that are below the achievable bearing speed. These limiting speeds are based on the following conditions:

- bearing preloaded, no external operating load
- 25% operating duration
- max. equilibrium temperature +50 °C.

If the speed is to be determined for a different application or higher speeds are necessary, a thermal balance analysis can be carried out. Please contact INA/FAG in this case.

Practically relevant limiting speed for applications in machine tools

Design of adjacent construction	Adjacent constructions (the shaft and housing) must be designed in accordance with the data in the dimension tables. The abutment diameters for the shaft and housing shoulders d _a and D _a must be in accordance with the dimension tables.
Caution!	D_a and d_a are recommended minimum abutment diameters. If these are not adhered to, the rib diameters d_1 , d_2 , D_1 , D_2 in the dimension table must be observed.
Mounting guidelines	
Caution!	Bearings should only be fitted and dismantled in accordance with the Fitting and Maintenance Manual TPI 100. This TPI is available upon request.
	During fitting of bearings, mounting forces should be applied only to the bearing ring to be fitted. Mounting forces must never be directed through the rolling elements or sealing rings.
	The characteristics of the bearings are only valid when used in combination with INA precision locknuts and the associated tightening torques given in the dimension table.
	Axial angular contact ball bearings are self-retaining and the individual bearing components are matched to each other. The inner rings must not be removed from the bearing during fitting and dismantling. If the inner rings have been separated from the bearing, please contact INA/FAG. Never use force to fit the bearing parts. This can lead to plastic deformations in the rolling element system that may render the bearing unusable.
Application of bearing preload	Axial angular contact ball bearings must be located clearance-free in the housing and on the shaft and must be axially preloaded during fitting. The axial preload force must be distributed evenly over the circumference in order to avoid deformation of the raceways.
Location of outer ring using a ring nut	Outer rings are axially clamped to the preload force given in the dimension table by means of a ring nut (not supplied by INA), <i>Figure 5</i> . The ring nut must be secured against loosening (e.g. using Loctite 638).
	Ring nuts with a runout of max. 5 μm apply the preload force evenly over the bearing rings and should therefore be used in preference to location by means of a cover.

1 Ring nut

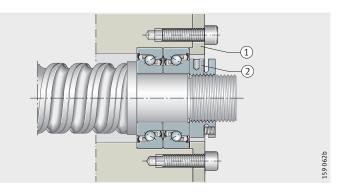
Figure 5 Two-bearing set in O arrangement, locknut, ring nut

Location of outer ring using a cover

Clamping of the outer rings using a cover and cap screws leads to deformation of the raceways, *Figure 6*. In order to minimise the deformation and achieve the calculated life:

- the cover should be designed with adequate rigidity
- the number of fixing screws should be selected in accordance with the load but at least four should be used
- the screws should be tightened in a cross-wise sequence in four stages (finger tight, 40%, 70%, 100% of M_{Δ}).

Caution! Observe the axial preload force in the dimension table. If other values are used, this will influence the bearing preload, bearing friction and heat generation in the bearing position.



Cover
 Locknut

Figure 6 Two-bearing set in O arrangement, locknut, cover

Location of inner ring

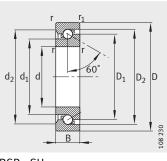
When clamping the inner rings using the recommended precision locknut, the tightening torques given in the dimension tables must be observed, *Figure 6*. The tightening torques for the individual bearing sizes are only valid for the INA precision locknuts listed.

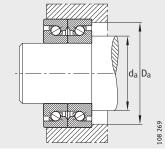
In order to counteract settling, it is recommended that the locknut should initially be tightened to twice the tightening torque M_A and then relieved of load again. It should only then be tightened again to the necessary tightening torque M_A .

Caution!If other suitable locknuts are used, the manufacturer's guidance on
calculation of the necessary tightening torque should be observed.Only locknuts with a minimum accuracy of the end face to the thread
of 5 μm should be used.

Single row axial angular contact ball bearings

Single direction Open or sealed on both sides





BSB..-SU

O arrangement

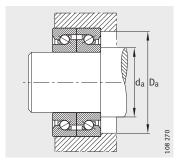
Dimension table · Dimensions in mm															
Designation	Mass	Dimensions									Mounting dimensions				
												for bearings in O arrangement		arings gement	
		d	d ₁	d ₂	D	D ₁	D ₂	В	r	r ₁	D _a H12	d _a h12	D _a H12	d _a h12	
	≈kg							-0,12	min.	min.					
BSB2047-SU															
BSB2047-SU-L055	0,13	20_0,005	26,8	34,5	47_0,006	37	43	15	1	0,6	43	27	43	26	
BSB2047-2Z-SU															
BSB2562-SU															
BSB2562-SU-L055	0,24	25 _{-0,005}	37,8	45,5	62 _{-0,007}	48	54	15	1	0,6	54	40	54	33	
BSB2562-2Z-SU															
BSB3062-SU															
BSB3062-SU-L055	0,22	30 _{-0,005}	37,8	45,5	62 _{-0,007}	48	54	15	1	0,6	54	40	54	37	
BSB3062-2Z-SU															
BSB3572-SU															
BSB3572-SU-L055	0,3	35 _{-0,006}	47,5	56	72 _{-0,007}	58,5	65,5	15	1	0,6	65	50	65	43	
BSB3572-2Z-SU															
BSB4072-SU															
BSB4072-SU-L055	0,26	40 _{-0,006}	47,5	56	72 _{-0,007}	58,5	65,5	15	1	0,6	65	50	65	47	
BSB4072-2Z-SU															

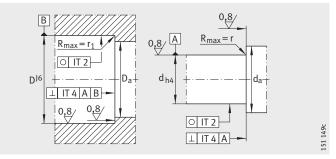
¹⁾ Valid for matched pair of bearings in O or X arrangement.

²⁾ Based on a preloaded bearing set comprising two sealed and greased bearings.

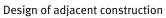
³⁾ Only valid in conjunction with INA precision locknuts.

⁴⁾ For applications in machine tools.





X arrangement



	Basic load ratings axial		Speeds		Rigidity ¹⁾ axial	Bearing frictional torque ²⁾	Run- out	Recommended INA locknut; to be ordered separately						
			Limiting speed	Practically relevant limiting speed ⁴⁾				for bearings in O arrangement; support for high inner ring		for bearings in X arrangement; support for low inner ring		Tightening torque ³⁾	Preload force	
-	dyn. C _a	stat. C _{0a}	n _G grease	n _G grease	c _{aL}	M _{RL}		shoulder		shoulder		M _A		
	N	Ν	min ⁻¹	min ⁻¹	N/µm	Nm	μm					Nm	Ν	
	26000	47 000	6 200	5 400	764	0,2	2	ZM20	AM20	ZM20	AM20	18	8 980	
	29 000	64 000	5 400	4 300	1001	0,25	2	ZMA25/45	AM25	ZM25	AM25	30	11 810	
	29 000	64 000	5 100	4 300	1034	0,25	2,5	ZM30	AM30	ZM30	AM30	32	11 080	
	37 000	89 000	4 700	3 850	1196	0,35	2,5	ZMA35/58	AM35	ZM35	AM35	50	15 220	
	37 000	89 000	4 700	3 850	1235	0,35	2,5	ZM40	AM40	ZM40	AM40	60	15 650	

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